

# TRANSVERSE SHEAR STIFFNESS OF T300/5208 GRAPHITE-EPOXY IN SIMPLE BENDING

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By: P.E. Sandorff, Staff Engineer Structures & Materials Laboratory



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### **FOREWORD**

This report describes work accomplished during 1980 and 1981 in the program, Experimental Mechanics. This program is funded by the Lockheed Independent Research and Development Program, and administered by the Structures and Materials Laboratory.

Acknowledgement is due K. N. Lauraitis and J. T. Ryder for their helpful discussions, and to R. C. Young, F. L. Imera, R. LaForce and F. M. Pickel for special care in specimen fabrication and testing.



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### **ABSTRACT**

Buckling and crippling of a structural element is governed by its elastic properties. Nevertheless, a complete definition of the elastic constants of a graphite/epoxy laminate, including the transverse shear moduli (important under high compressive stress), has never been accomplished. For this purpose, tension, compression, and bending tests were conducted on square-section specimens cut 0-degrees and 90-degrees to the fiber axis from 64-ply unidirectional T300/5208 laminate. Axial test specimens were instrumented with tee gages to obtain data on extensional moduli and Poisson's ratios. Bending deflections were measured with high precision and analyzed using a rigorous finite difference solution for stress distribution to derive the shear moduli along with the extensional moduli acting in bending. Redundant determinations were obtained for the nine elastic constants of Hooke's law as generalized for specially orthotropic material.

Extensional moduli obtained under axial load and in bending agreed with each other to within one percent. A self-consistent set of properties was determined which were generally in accordance with published data, and which established the material as being transvers by isotropic and conforming to the linear theory of elasticity. The value obtained for the shear modulus  $G_{12}$  (= $G_{13}$ ) was 4500 MPa (0.65 Msi), some twenty percent lower than that generally accepted on the basis of  $\pm 45^{\circ}$  tension test data. The value of the transverse shear modulus  $G_{23}$  was 3650 MPa (0.53 Msi), in accordance with published results of ultrasonic tests.



### INTRODUCTION

In the design and analysis of metal aircraft structures which buckle under compressive loads, and those which deform under transverse bending load. it is seldom necessary to consider the effects of shearing deformation. The transverse shear stiffness of graphite-epoxy composite laminate, however, is no more than a fifth that of aluminum alloy, and the adverse effects of shearing deformation will require consideration in many practical applications of advanced design. Reference 1, for example, indicates a reduction in compressive buckling stress of fifty percent for stiffener elements designed for high stress (width to thickness ratio of 10). An evaluation of the transverse shear modulus of this material to a reasonably high confidence level is therefore necessary for applying it efficiently in aircraft construction.

The elastic properties of composite laminate are usually represented as specially orthotropic. In this analytic model, unidirectional laminate has three shear moduli: the in-plane shear modulus  $G_{12}$ , and two transverse moduli  $G_{13}$  and  $G_{23}$ . There are several direct, relatively simple test methods for evaluating  $G_{12}$ , but very few for the transverse moduli. A selected bibliography is presented in References 2 through 15.

The most commonly used method for evaluating the in-plane shear modulus of composite laminate is the  $\pm$  45° tensile test (Reference 5). In this test, a tee or rosette strain gage provides bi-directional stress-strain data; two-dimensional transformation of stresses and strains identifies shear stress vs. shear strain in the plane of the laminate. While this test is concerned only with response in the plane of the laminate, the material is usually assumed to be transversely isotropic on the basis of its structure, in which case  $G_{13}$ = $G_{12}$ . Investigations have demonstrated this method to be equivalent (probably to the limitations of experimental control) to the cross-sandwich beam, thin



tube in torsion, ten-deg off axis, panel shear, and rail shear tests (References 3, 6, 8). Tests of T300/5208 graphite/epoxy of 0.62-0.67 fiber volume fraction by the  $\pm 45^{\circ}$  tension method have indicated  $G_{12}=0.74-0.80$  Msi (References 9,18). This value is generally understood to be the initial slope of the stress-strain relation. A possibly significant aspect of all the test methods in this category is that they usually furnish decidedly non-linear shear stress-strain curves (for example, see References 3, 6, 8, 9, and 16).

The complete set of material elastic constants can be evaluated from the speed of acoustic wave propagation in ultrasonic tests of specimens of special geometries. Such experiments indicate  $G_{12}$  and  $G_{13}$  to be about thirty percent higher than obtained with the  $\pm$  45° static tension test; Reference 7, for example, gives 1.03 Msi for this property, with  $G_{23}=0.527$  Msi. Engineers hesitate to use such values in design because a one-to-one relationship between microscopic dynamic oscillations and macroscopic static deformation has not been demonstrated for graphite-epoxy; indeed, even in isotropic metals, the two techniques usually provide slightly different results.

Since one of the engineering uses of the shear modulus is to improve the analytic prediction of buckling phenomena, a practical approach is to test simple buckling specimens, determine the reductions in critical load due to shear effects, and back-figure to derive applicable values of the shear moduli. This method was used to analyze some 1800 column test data in Reference 9. For T300/5208 laminate, this approach indicated room temperature values of  $G_{13} = 0.30$  Msi and  $G_{23} = 0.045$  Msi. Besides being unexpectedly low, these values conflict with the assumption of transverse isotropy.

In the past few years the Air Force Materials Laboratory has developed a direct test method for evaluating two of the shear moduli. This method utilizes a half-inch diameter, six-inch long cyclinder cut 90 degrees to the fiber direction from thick unidirectional laminate. Shear strain gages are mounted on the surface at 0 degrees and 90 degrees to the fiber and the cylinder is loaded in torsion (Reference  $\frac{12}{5}$ ). Tests of T300/5208 of fiber volume fraction 0.63 reported in Reference  $\frac{13}{5}$  give values of 0.866 and 0.506 Msi for  $G_{12}$  and  $G_{23}$ , respectively.



Deserving of mention for its simplicity and low cost, a method known as the Iosipescu shear test has recently been applied to composite laminates (Reference 15). This test uses a specimen only slightly larger and harder to fabricate than that for the ASTM D 2344 short beam shear test, and it makes material characterization possible for both shear stiffness and strength in all three planes. The method must be confirmed by test results obtained under loading conditions realistic for structural applications, since (as with most methods) the geometry is unusual and the purity of the stress condition might be questioned.

A method that is appealing from a practical standpoint determines the shear stiffness from simple flexure tests in accordance with accepted handbook-type formulas for bending and shear deflection (References 10 and 14). Application of this method in Reference 10 indicated  $G_{13}$  of 16-ply T300/5208 laminate to be 0.40 Msi. This method suffers from the crudeness of the handbook approximation made for shear deflection, as well as from the difficulty of achieving adequate experimental precision.

The basis for an accurate analysis of flexure test data was established, in Reference 19, by the development of a computer-aided solution for the plane stress representation of a simple 3-point beam of orthotropic material. This solution provides a precise determination of the complete normal and shear stress distribution, and so permits accurate prediction of the bending and the shearing components of elastic deflection. This predictive calculation can be applied to determine the values of the elastic moduli which best fit redundant sets of deflection data obtained in flexure tests at different spans. The investigation described in this report was planned to make use of this approach, in order to evaluate the elastic constants of graphite/epoxy laminate under loading conditions representative of typical structural applications.



### **OBJECTIVE**

The express purpose of this investigation was to evaluate the transverse shearing stiffness of T300/5208 unidirectional laminate when subjected to simple bending.

In the completion of this objective, answers to the following additional questions were also sought:

- o Which if any of the existing test methods for shear modulus could be confirmed.
- What causes the disagreement among values obtained by different test methods.
- o Can the material be considered transversely isotropic or not.
- o Is more fundamental study required on the nature of the material and the elastic model used to represent its response.



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### **APPROACH**

Four elastic moduli govern the plane stress or plane strain response of orthotropic material. In the x-z plane these are the two extensional moduli  $E_{\chi\chi}$  and  $E_{\chi\chi}$ , the transverse shear modulus  $G_{\chi\chi}$ , and the Poisson's ratio  $v_{\chi\chi}$ . The bending stiffness of a flexure specimen is a strong function of  $E_{\chi\chi}$ , a weak function of  $G_{\chi\chi}$ , and almost independent of  $E_{\chi\chi}$  and  $v_{\chi\chi}$ . Therefore, two flexure tests obtained on the same beam at different spans provide sufficient information to evaluate  $E_{\chi\chi}$  and  $G_{\chi\chi}$  to good precision. To do this, however, it is necessary to relate observed deflections to the moduli through an accurate representation of the stress distribution.

The two-dimensional stress distribution for an orthotropic beam can be solved quickly and to any desired degree of accuracy by the computer-aided relaxation technique described in Reference 8. This is a rigorous finite difference analysis which derives values of the stress function over a matrix of interior points to suit a specified set of boundary conditions. Once the stress function matrix is determined, stresses are readily computed, and the strains and deflections follow upon selection of the moduli.

The investigation was planned to take advantage of conventional axial load tests to identify the extensional moduli and the Poisson's ratios of a sample unidirectional laminate. Identical square section specimens cut at 0 degrees and 90 degrees to the fiber axis from 64-ply graphite-epoxy were used for all tests. A number were tested under axial load; these were instrumented with strain gages. Replicate specimens were tested in three-point bending, the stiffness being determined by measurement of central deflection. Each bending specimen provided data in two planes of loading under identical test conditions. The unreduced test data therefore furnished qualitative information regarding transverse isotropy and the relative magnitudes of  $G_{13}$  and  $G_{23}$ . The experimental approach is illustrated schematically in Figure 1.



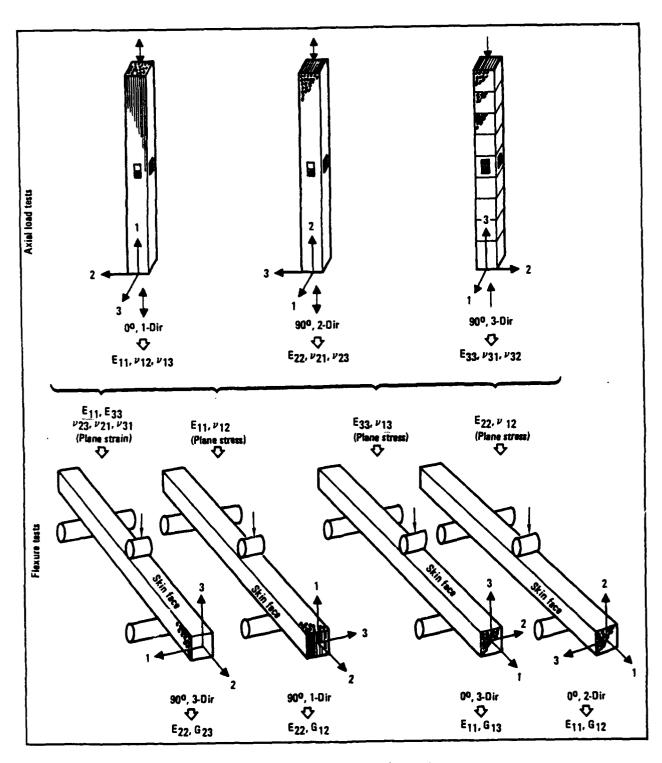


Figure 1. - Test Plan Schematic

Deflection measurements made in the flexure tests included deformation occurring in the test fixture and Hertzian-like bearing deformation of the surface of the beam specimen. These effects comprised an error that was significant for tests at very short span. To provide correctional data, supplementary tests were conducted on specimens loaded as a block in bearing, and on the test fixture with the specimen removed. The correction for bearing deformation was derived from the block-bearing test data with the aid of two additional computer-aided relaxation analyses, by which the Hertzian deformation occurring in the block specimen was related to that occurring in the beam.

Reduction of the flexure test data was achieved by applying the analysis for beam deflection to each test span and condition, using a trial—and—correction procedure to obtain values of the moduli which best fit the multiple test points. The procedure is illustrated by the flow chart of Figure 2.

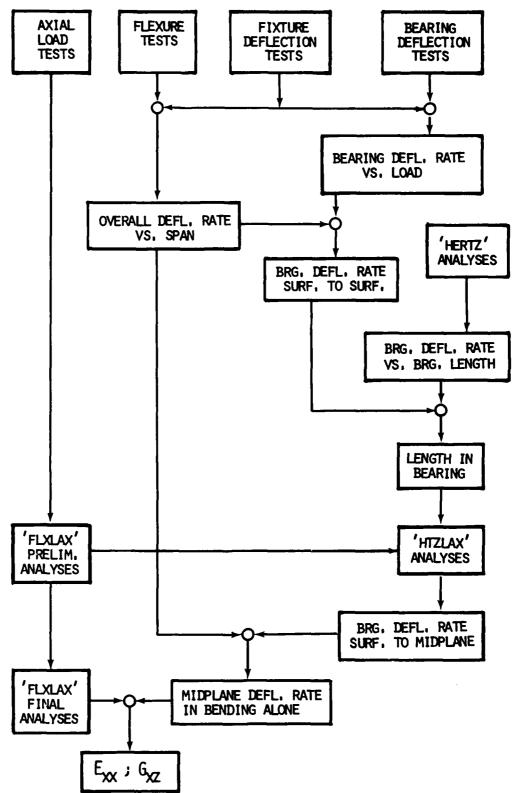


Figure 2 - Flow Chart, Test and Analysis Procedure

### **SPECIMENS**

To furnish stock for test specimens, a 64-ply unidirectional panel was fabricated of T300/5208 low-resin content (35 %) prepreg tape. The layup with symmetrical bleeder plies was placed under vacuum in a press for the standard cure cycle (45 minutes at 132°C followed by 2 hours at 179°C); no post-cure was applied.

To remove resin-rich surface material, a grinding operation using water coolant was performed on both surfaces of the panel. Square specimens of ten-inch length were then cut 0 degrees and 90 degrees to the laminate axis. Final grinding reduced all sections to a uniform 10.59 x 10.59 mm (0.269 x 0.269 inch). Section dimensions were measured to  $\pm 0.0001$  inch at stations spaced every inch along the length.

Photomicrography of two specimens after test showed a high degree of homogeneity, although traces of the ply boundaries indicated irregular distortions from the plane. Photomicrographs are presented in the Appendix.

Resin analysis performed per ASTM D3171 on samples taken from the finished specimens indicated a laminate specific gravity of 1.61 and a fiber volume fraction of 68.6 percent. The specimens were dried at  $66\,^{\circ}\text{C}$  (150°F) in vacuo for two months prior to test, a procedure estimated to reduce the moisture content to less than 0.1 percent.

Three specimens of each orientation were used for axial tests to obtain extensional properties at 0 degrees and 90 degrees to the fiber direction. The problem of applying test load in the thickness (3) direction was met by cutting short lengths from specimens tested under axial load and reassembling the short sections by adhesive bonding to form 3-tier and 5-tier compression specimens.



Additionally, to obtain specimens for axial load in the 3-direction which were geometrically similar to the others, cubical elements were cut from a fourth 90-degree specimen, reoriented through 90 degrees, and bond-assembled as indicated in Figure 1. These assembled specimens were intended for test under axial tension, but bond failure prevented tension loading of useful magnitude. One of these specimens, which incorporated seventeen cubical elements, was salvaged for axial compression testing by squaring the ends.

Three additional specimens each, of  $0^{\circ}$  and  $90^{\circ}$  orientation, were selected for flexure tests. When supported for beam tests the central load was always applied at the specimen midstation, with excess specimen length extending symmetrically beyond the beam supports. Maximum variation in measured section dimensions over the test span for any of these specimens was less than 0.3 percent.

### EXPERIMENTAL PROCEDURE

### Axial Tests

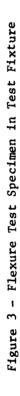
Axial test specimens were instrumented with tee foil gages centrally mounted on all four faces and tested in a universal testing machine. Gage output vs. load data were automatically recorded and reduced with the Rye Canyon centralized digital data acquisition system, which has a demonstrated overall accuracy better than 0.05% of selected range. Tension loading not exceeding 0.003 strain was first applied using Templin-type mechanical grips. The specimens were then reduced to approximately 10 cm (4 inch) length, the ends squared, and compression loads applied between flat, adjustable platens. In compression tests of built-up specimens in the thickness (3) direction, the element bearing the gages was always the central member of a stack of identical and identically oriented elements, in order to minimize end effects.

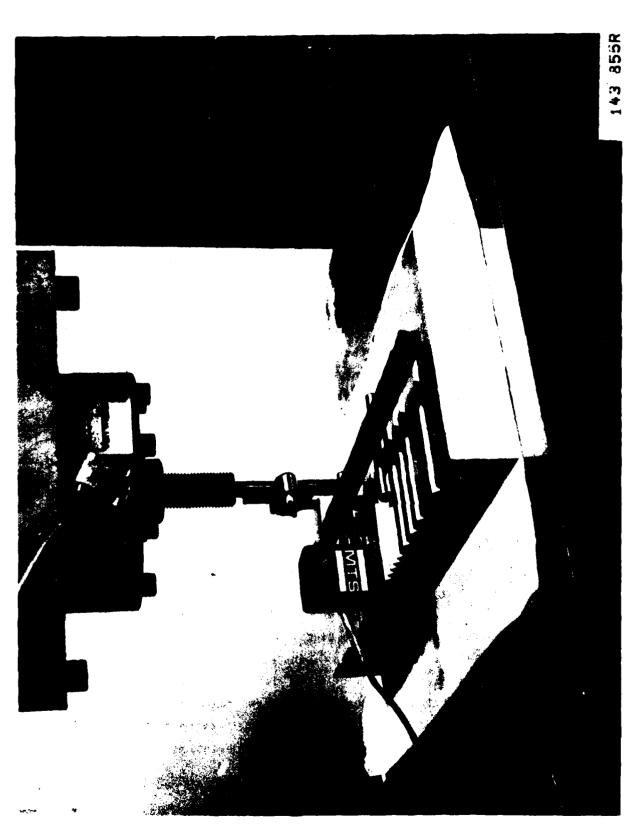
Representative stress-strain curves obtained in these tests are presented in the Appendix.

### Flexural Tests

Flexural tests were conducted in the special fixture shown in Figure 3. Load was applied and reacted through hard steel surfaces of 3.175 mm (0.125 inch) radius. The width of the loading nose matched that of the specimen. Deflection was measured between the loading nose and the fixture base with a clip gage (MTS 632.02B01). The Rye Canyon centralized digital data system was used to acquire, record, and reduce load-deflection data. End-to-end calibration of the deflection measurement and recording system against a certified Templin extensometer calibrator indicated overall accuracy and linearity of  $\pm$  1.3 micrometer (0.00005 inch) maximum deviation over a range of 2.5 mm (0.10 inch).







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Tests were conducted at eight different spans on each specimen, providing replicate data from three specimens for each of the four beam orientations. Test span lengths had to be short enough so that shear deformation was large compared with the expected error in measurement, yet not so short that the error caused by local bearing deformation became intolerable. Maximum applied loading corresponded to a beam theory strain Mc/EI of about 0.004. Representative deflection vs. load curves obtained in these tests are reproduced in the Appendix. Except for initial softness, which may be attributed to bearing on asperities and possibly to slight accommodation in torsion, load-deflection relations were quite linear.

The measured load vs. deflection included load carried by the clip gage as well as that on the test specimen. A test was conducted on the clip gage alone, measuring deflection with a dial gage, to evaluate this quantity. The results, which are included in the Appendix, provide a correction which was important for the longest test spans.

A supplementary test was performed to evaluate the error introduced into the flexure test data by deformations which occurred in components of the test fixture included between the gage points. For this test the loading nose was brought into contact with the base of the fixture, and the load-deflection characteristics recorded as before. The resulting curve is reproduced in the Appendix. (These measurements provided only an approximation to the behavior in the flexure test because some deformation occurred in the base under the loading nose instead of at the beam supports).

Tests were also conducted with load applied to representative specimens as in the flexure tests but with the specimens resting directly on the flat base of the test fixture, in order measure specimen bearing deformation. Deflection vs. load curves obtained in these tests are included in the Appendix.



### TEST RESULTS AND ANALYSIS

### **Axial Tests**

Compliance data under axial stress were derived by manually evaluating the slopes of the stress-strain curves furnished by the data processing system. This is an operation that produces repeatable values within one half to one percent if the slopes are truly linear. Linearity existed over the major portion of the curve for all tests except those for tension in the fiber direction, in which stiffness increased slightly with load. In these cases secant data were obtained over several ranges.

Corrections for strain gage transverse sensitivity were applied after evaluation of the slopes. Tabulations of the material compliance coefficients, obtained by averaging data from back-to-back strain gages, are recorded in the Appendix. Mean values and coefficients of variation are presented in Table 1.

With only moderate exception, these data are self-consistent and fit the usually assumed model which displays symmetric coupling effects ( $S_{12} = S_{21}$ ,  $S_{13} = S_{31}$ ,  $S_{23} = S_{32}$ ) and transverse isotropy ( $S_{22} = S_{33}$ ,  $S_{12} = S_{13}$ ). A set of coefficients for this model, obtained by averaging all determinations bearing on any particular value, is as follows (units of  $\mu e/MPa$  ( $\mu e/psi$ ):

Corresponding values of the engineeering constants are listed in Table 2.



TABLE 1 SUMMARY OF COMPLIANCE COEFFICIENTS FROM AXIAL TESTS

PROPERTY DIRECTION Parallel to fiber Sil (1-dir) S21 S21	ATA											
		STRESS RANGE MPa (ksi)	z	MF µe/MPa	MEAN ue/MPa (µe/psi)	C.V.	STRES MPa	STRESS RANGE MPa (ks1)	Z	M µe/MPa	MEAN ue/MPa (µe/ps1)	C.V.
		70-280(10-40)	80	69.9	6.69 (0.0461) 1.4	1.4	2-80	5-80 (1-11)	9	7.15	7.15 (0.0493)	1.6
S <sub>21</sub>		70-280(10-40)	4	- 2.18	2.18 (-0.0150) 2.7	2.7	2-80	5-80 (1-11)	ო	- 2.19	- 2.19 (-0.0151)	3.6
10	= -	70-280(10-40)	4	- 2.13	2.13 (-0.0147) 3.0	3.0	2-80	(1-11)	<del>ن</del>	- 2.15	(-0.0148)	9.9
In-plane trans. \$22		(9-0) 07-0	œ	93,3	93.3 (0.643)	1.2	5-20	5-20 (1-3)	9	91.1	(0.628)	1.9
S <sub>12</sub>	. ,	0-40 (0-6)	4 4	- 2.26 -44.5	- 2.26 (-0.0156) 1.8 -44.5 (-0.307) 0.7	1.8	5-20	(1-3) (1-3)	ო ო	- 2.12 -43.1	(-0.0146) (-0.297)	5.9
Thickness dir. 833		ı	0		ı	1 -	5-13(	5-130 (1-19)	0,	97.4	(0.671)	1.4
(3-41r) <sup>S</sup> 13		1 1	0 0		1 1	1 1	5-13(	5-130 (1-19) 5-130 (1-19)	4 2	- 2.42	(-0.0167) (-0.346)	7.2

### TABLE 2 RESULTS OF AXIAL LOAD TESTS

	n	Value	Coef Var'n
Extensional Moduli:			
E <sub>11</sub>	14	21.0 Msi	2.1
E <sub>22</sub> , E <sub>33</sub>	27	1.54 <b>Msi</b>	2.9
Poisson's Ratios:			
ν <sub>12</sub> , ν <sub>13</sub>	14	0.31	3.8
ν <sub>23</sub> , ν <sub>32</sub>	13	0.49	7.3
ν <sub>21</sub> , ν <sub>31</sub> (= ν <sub>12</sub> E <sub>22</sub> /Ε <sub>11</sub> )	13	0.023	8.8

### Flexure Tests

Recorded test results were provided by computer in the form of load-deflection curves, which were analyzed manually to obtain values of slope. The resultant deflection rate data ( $\mu$  in./1b) and the corresponding range of loading over which the behavior was essentially linear are listed in Table 3.

The correctional data also were reduced to terms of deflection rate. The correction for clip gage load was incorporated by combining it with the average deflection rate obtained from the three specimens tested under each of the thirty-two test conditions. These results are included in Table 3. Bearing deformation effects were non-linear in nature, and reduction of the data produced the deflection rate vs. load relations presented in Figure 4. A preliminary analysis of the flexure test data was required before the correction for bearing deformation could be applied.

The finite difference relaxation technique of Reference 19, which was used in analyzing the flexure test data, is based on the stress function relation for orthotropic material:

$$K_{z} = \frac{\partial^{4} \phi}{\partial x^{4}} + 2 \frac{\partial^{4} \phi}{\partial x^{2} \partial z^{2}} + K_{x} = \frac{\partial^{4} \phi}{\partial z^{4}} = 0$$
 (1)

Here, for plane strain,

$$K_{x} = \frac{(1-v_{xy}^{2} E_{yy}/E_{xx})}{E_{xx}^{2G}_{xz} - v_{xz} - v_{xy} v_{zy} E_{yy}/E_{zz}}$$
(2)

$$K_z = K_x \frac{E_{xx} (1 - v_{zy}^2 E_{yy}/E_{zz})}{E_{zz} (1 - v_{xy}^2 E_{yy}/E_{xx})}$$
 (3)

Plane stress is expressed by placing  $E_{vv}$  =0.



TABLE 3. RESULTS OF BENDING TESTS

			FLEXURE	FLEXURE TEST DAT	I'A					BEARING DEFL. RAIES ( HIN./LB)	ATES ( + IN.	/LB)	.,410	BEARING	SEARING CORR. (FIG.	(5. 5)	NET DEFL.
CRIES	BEAM CRIENTATION	SPAN L (1N.)	LOAD RANGE (LB)	, OV.	NO. S	COR NO. 6 AVE	CORRECTED	OVERALL	A FIXTURE ALONE	(B, NET IN SPECIMEN	OVERALL	FIXTURE (C. )	© NET IN SPECIMEN	AT	T AT TER SUPP'T	TOTAL A + B + C/2	TEST DATA REDUCED ( H IN, /LB)
•	3-dir	1.920 1.520 2.024 2.520 3.020 4.030 5.017	200-600 150-400 80-220 50-200 50-150 40-130 20-120	13.3 21.4 34.6 55.0 82.5 170 312 520	13.4 34.7 55.0 84.0 172 313	34.1 55.0 83.6 1173 317	13.4 21.5 34.5 55.1 83.6 173 317	5.50	1.5 1.6 2.0 2.1 2.2 2.3 2.3 2.6	6.00 0.44 0.44 0.44 0.44 0.44 0.44 0.44	2.9 7.3 7.9 8.3 9.3	2.5 2.5 2.5 2.5 2.6 6.7	11444000	2.17	1,1 8 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	2444.0.0.0.0.0 604.0.0	20.5 50.5 78.6 78.6 168 111 523
°o	2-dir	1.020 1.520 2.024 2.024 2.5.0 4.020 5.017 6.020	200-600 150-400 80-220 50-200 50-150 40-130 30-120	13.4 22.8 34.6 55.5 84.0 171 313	13.4 21.3 35.0 35.0 55.5 171 171 314 528	24.4 54.4 81.0 315 524	13.4 22.1 34.7 55.1 63.3 173 317	5.5 5.6 6.2 6.6 6.6 7.0	-1.5 2.0 2.1 2.1 2.3 2.3 2.3	6.0044444 0.0044444	2.7.7.7.9.9.9.9.9.9.9.9.9.9.9.9.9.9.9.9.	22.22.22.23.23.23.24.26.28.26.26.26.26.26.26.26.26.26.26.26.26.26.	4444886	, , , , , , , , , , , , , , , , , , ,	11 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	2444000 604	-8 -18 31 50.5 78.4 168 168 311
<b>\$</b>	3-dir	1.020 1.520 2.024 2.520 3.020 4.020 5.017 6.020	25-75 20-40 15-35 15-35 5-20 1-20 1-10	48.5 126 276 508 847 1920 3570 5700	47.5 127 272 510 857 1920	48.4 12.5 27.2 51.5 51.5 86.5	48.2 126 276 519 878 2030 3980 6790	9.5 10.8 11.4 13.4 15 ~15 ~15	244464444 44444444444444444444444444444	8.9 8.5 110 111 112 112	4.2.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.1.	111 0.000	28.5 10.1 10.1 10.1 10.1 10.1	4 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	กร จำตัดผลตลด	9.80 t 80 t	39.0 117.5 269 511 871 2020 3970 6780
• 06	141	1.020 1.520 2.024 2.520 3.020 4.020 5.017	30-70 20-40 10-40 5-25 5-20 4-20 2-16	43.8 121 272 213 857 1940 3610 5700	42.0 117 266 505 837 837 1920 3610	42.8 121 271 508 855 -	42.9 120 272 516 871 2040 4030 6800	8.4.2.7.7.4.8	7,7,2,E,8,4 \$\dot{2},2	3622225	1   1   1   1   1   1   1   1   1   1	244439677	2000.52222	222222		w.4.82.80 c 88	39.1 115.3 267 510 865 2030 4020 6790

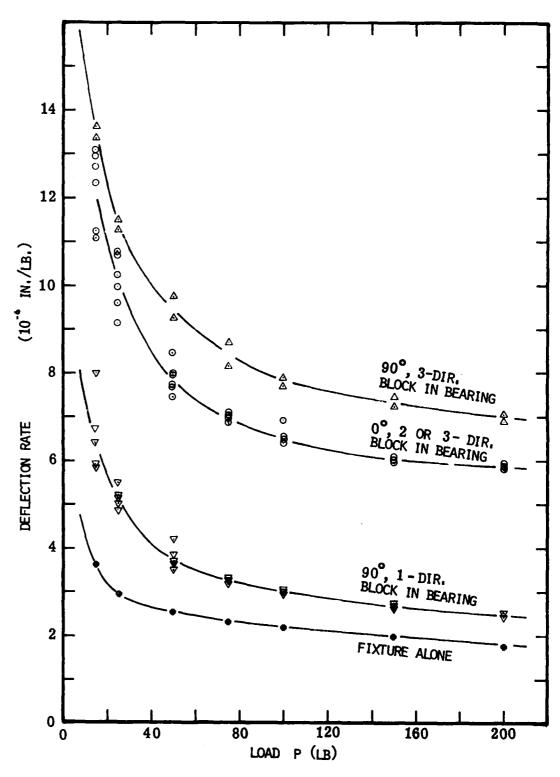


Figure 4 - Experimental Deflection Rates (Slopes of Recorded Load - Deflection Curves)

The stress function solution was applied to treat the central portion of the test specimen out to a distance 2h (twice the beam depth) from the point of load application. Deflection of the remainder of the span was calculated by beam theory. The net of point values was spaced h/10 in the spanwise (x) direction and h/20 in the depth (z) direction. The model is shown schematically in Figure 5. Preliminary studies showed that this analysis provided the desired precision in deflection calculation.

Preliminary solutions of the stress function matrix were derived for each of the flexure test conditions, using elastic constants determined in the axial tests and assuming values of the shear moduli. The case of beam load in the 2-3 plane (obtained in the 90-degree beam with fibers oriented transverse to the load) was analyzed as plane strain, the others as plane stress. Best-fit values of  $E_{\chi\chi}$  and  $G_{\chi\chi}$  were then derived by placing

$$(dw_{test}/dP)_{i} = a (dw_{b}/dP)_{i} + b(dw_{s}/dP)_{i}$$
 (4)

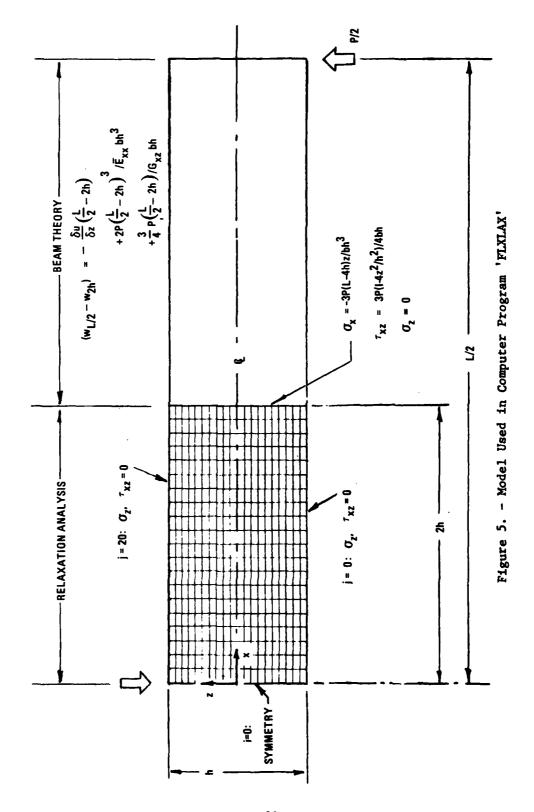
Here  $dw_b/dP$  and  $dw_s/dP$  are deflection rates attributable to bending stress and to shearing stress, respectively, calculated at the beam midplane; a and b are error factors. Defining, for simplification,

$$x = \frac{dw_s/dP}{dw_h/dP} ; y = \frac{dw_{test}/dP}{dw_h/dP} (5)$$

Then

$$y_{i} = a + bx_{i}$$
 (6)

and a and b may be derived by linear regression of y on x. If bending and shear deformation are assumed to occur independently of each other, a and b become correction factors for the moduli  $\mathbf{E}_{\mathbf{XX}}$  and  $\mathbf{G}_{\mathbf{XZ}}$ . The error in this asssumption becomes negligible if the correction is small, making iterative improvement of the relaxation solution a practical measure.



Correction of the flexure test data was then made for bearing deformations occurring between the surface at which load was applied and the midplane of Analysis was required to derive this correction quantity from measurements taken surface-to-surface on a block which carried no bending stress. General solutions for the two cases (viz., block in bearing supported at the opposite surface, and section of a beam supported by moment and shear at the ends) were obtained by extensions of the stress function relaxation analysis described above which used closer grid spacing. For each of the flexure test conditions, a set of specific solutions for deformation under the two support conditions was derived for a range of values of the width over which the central load was distributed. The block-bearing test data could then be used to evaluate a load distribution parameter for each condition. By assuming this parameter applied also to the beam-supported bearing problem, the applicable solution was identified and the bearing deformation occurring between the loading nose and the beam midplane was evaluated.

This analytic procedure is considered only an approximation, which is especially limited for short spans and high loads, but adequate as a correction procedure. To facilitate the correction process, the analytic solutions are arranged in momographic fashion in Figure 6. Entry to the lower set of curves with the block-bearing deflection rate, obtained from Figure 4 at the median of the load range of interest, lead via the upper curves to the surface-to-midplane deflection rate in the flexure test. Data derived by this method for the loading station, and (under half the beam load) for the support station, were applied to correct the flexure data in Table 3. Also included in Table 3 are corrections for deflection occurring in the fixture, taken from Figure 4.

With the flexure test data refined, additional iterations of the solution for  $E_{\chi\chi}$  and  $G_{\chi\chi}$  were performed until best-fit values to the eight test points of each set were obtained. Variation in the Poisson's ratios from the values determined in the extensional tests was not investigated, other than to



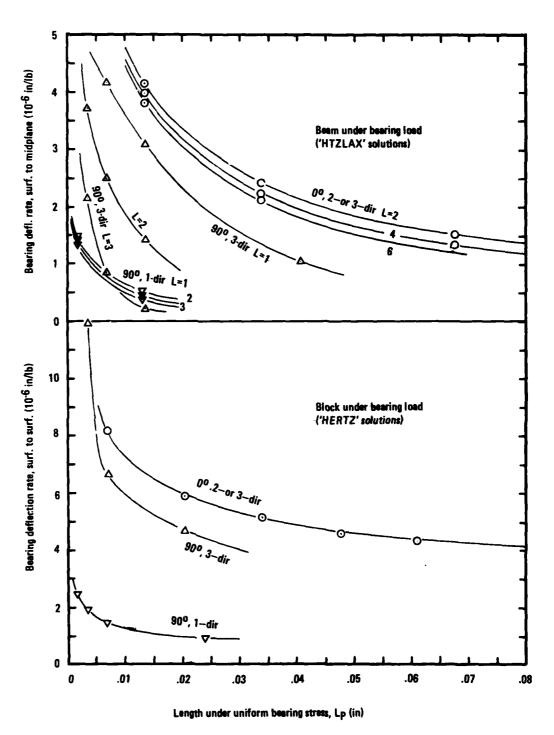


Figure 6. - Nomograph for Converting Block Bearing Deflection Data to Surface-to-Midplane Bearing Deflection of Beam

demonstrate the effect to be too small to utilize. The results, which provide at least one and in some cases two determinations of the extensional and the shear moduli, are presented in Table 4 and summarized in Table 5. Extensional and shear moduli derived from beams of different fiber orientation are seen to be in good agreement, and agreement of the two transverse shear moduli  $G_{12}$  and  $G_{13}$  (from  $0^{\circ}$  and  $90^{\circ}$  beams, respectively) again confirms the material as being transversely isotropic. Furthermore, the transverse shear modulus  $G_{23}$  is in almost exact agreement with the value expected by transverse isotropy, namely,  $G_{23} = E_{22}/2(1+v_{23})$  or .52 Msi.

The fit to the test data achieved by iterative refinement of the relaxation solution is indicated in Figure 7. While this plot demonstrates that the computation provides an exceptionally good representation of total beam deflection, it is misleading with regard to the precision of the shear modulus determination. This process is better characterized by the plot of final values of the deflection rate ratios presented in Figure 8.



# TABLE 4. ANALYSIS OF FLEXURE DATA

ysis			r=0.9999	a=0.9973	b=0.9965	E11=21.26	G13=0 452	1				r=0.9991	a=1.0068	b=1.0010	E11=20.96	612=0.649		4U 45		r=0.9986	a=1.0151	b=0.9709	E22=1.527	623=0.536		200	18. 18. (1. (1. (1. (1. (1. (1. (1. (1. (1. (1	r=0.9876	a=0.9990	b=0.9665	E22=1.552	612=0.677			
Fit Analysis	YC: )	6.4767	2.8330	1.9487	1.5665	1.3721	1 2010	1 1249	1.0987	1 13.	6.4438	2.9777	1.9487	1.5557	1.3594	1.1924	1.1170	1.0943	1.3313	1.1464	1.0910	1.0644	1.0484	1.0260	1.0352	1.0295	1.2456	1.0851	1.0524	1.0362	1.0187	1.0110	1.0288	1.0127	
Best F	XCO	5.5135	1.7986	0.9306	0.5812	0.3964	4010	6 1398	0.0976		5.4850	1.7852	0.9306	0.5771	0.3937	0.2178	0.1388	8960.0	0.3227	0.1409	0.0200	0.0509	0.0354	0.0200	0.0129	0.0000	0.2456	0.1103	0.0621	0.0400	0.0279	0.0158	0.0101	0.007	
	w(tot)	8.0	16.8	30.7	51.0	80.0	170.4	7,50	522.5		3.1	16.8	30.7	51.2	80.4	171.6	317.1	526.2	38.7	116.9	266.0	504.5	860.2	2008.2	3884.4	6644.9	39.1	118.0	269.4	511.9	872.8	2039.7	3947.3	6/52.2	
ons	3	6.81	10.79	14.80	13.74	22.71	20 69	48 64	46.44		6.81	10.79	14.80	18.73	22.71	30.69	38.64	46.44	9.45	14.45	19.47	24.42	29.41	39,38	49.33	59.19	7.71	11.72	15.75	19.71	23.79	31.68	39.63	47.52	
Relaxation Solutions	<b>q</b> .::	1.24	9.00	15.91	32.24	57.28	4 4 99	276 48	476.03		1.24	6.04	16.03	32.46	57.67	140.90	278.42	479.76	29.30	102.49	246.57	480.07	830.77	1968.82	3835.11	6585.70	31.39	106.26	253.70	492.16	849.14	2007.98	3907.63	6/04:/1	
Relaxa	Assumptions		Plane Stress	Exx=21.2 Msi	Ezz=1,55 Msi								Exx=21.1 Msi	Ezz=1.55 Msi		Uxz=).31			Plane Strain	Exx=1.55 Msi	Eyy=21.0 Msi		6xz=0.52 Msi	Uxz=0.49	Uxy=0.023	Uzy=0.023		Plane Stress	Exx=1.55 Msi	Ezz=21.2 Msi	Gxz=0.65 Msi	Uxz=0.023			
	w'(tst)	89	17	31	50.5	78.6	675	444	503		8	18	31	50.5	78.4	168	311	525	39.0	117.5	269	511	871	2020	3970	6780	39.1	115.3	267	510	865	2030	4020	9679	
	Span	1.020	1.520	2.024	2.520	3.020	000	010	6.005		1.020	1.520	2.024	2.520	3.020	4.020	5.017	9.002	1.020	1.520	2.024	2.520	3.020	4.020	5.017	9.005	1.020	1.520	2.024	2.520	3.020	4.020	5.017	6.005	
Test Results	Width	0.2691	0.2691	0.2691	0.2691	0.2691	1070 0	0 2404	0 2691		0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	0.2692	0.2692	0.2692	0.2692	0.2692	0.2692	0.2692	0.2692	0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	0.2695	
Test A	Depth	0.2695	0.2695	0.2695	0.2695	0.2695	1070	0 2605	0 2695	7,07.0	0.2691	0.2691	0.2691	0.2691	0.2691				0.2695								0.2682	0.2682					.2682	0.2682	
	Beam Orient.	0-deg, 3-dir	0-deg, 3-dir	0-deg.3-dir	0-deg.3-dir	0-deg .3-dir	7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7	110 C (690 C	O-dept of	1000000	0-deg, 2-dir	70-deg, 3-dir	90-deg. 3-dir	20-deg , 3-dir	90-deg, 3-dir	20-deg, 3-dir	90-deg, 3-dir	90-deg, 3-dir	70-deg, 3-dir	90-deg,1-dir	90-deg, 1-dir	90-deg, 1-dir	70-deg, 1-dir	90-deg, 1-dir			90-deg,1-dir								

Beam dimensions in inches. Deflection rate w'=dw/dP, win/lb. w'b=bending component, w's=shear component. Best fit analysis by linear regression of Y on X; X=w's/w'b; Y=w'tst/w'b; Y=a+bX

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TABLE 5. SUMMARY OF FLEXURE TEST RESULTS (Extensional and Shear Moduli in Msi)

Test Condition	0 <sup>0</sup> , 3-dir	0 <sup>0</sup> , 2-dir	90 <sup>0</sup> , 3-dir	90 <sup>0</sup> , 1-dir
N	22	22	20	21
E <sub>11</sub>	21.26	20.96	-	-
E <sub>22</sub>	-	-	1.527	1.552
E <sub>33</sub>	-	-	_	-
G <sub>12</sub>	-	-	-	0.677
G <sub>13</sub>	0.652	0.649	-	-
G <sub>23</sub>	_	_	0.536	-

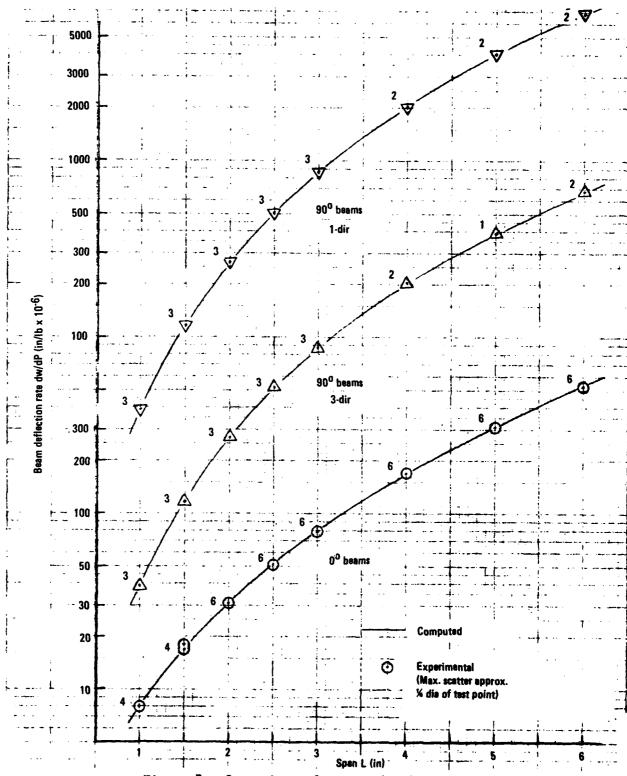


Figure 7. Comparison of Computed Solutions for Beam Deflection with Test Data



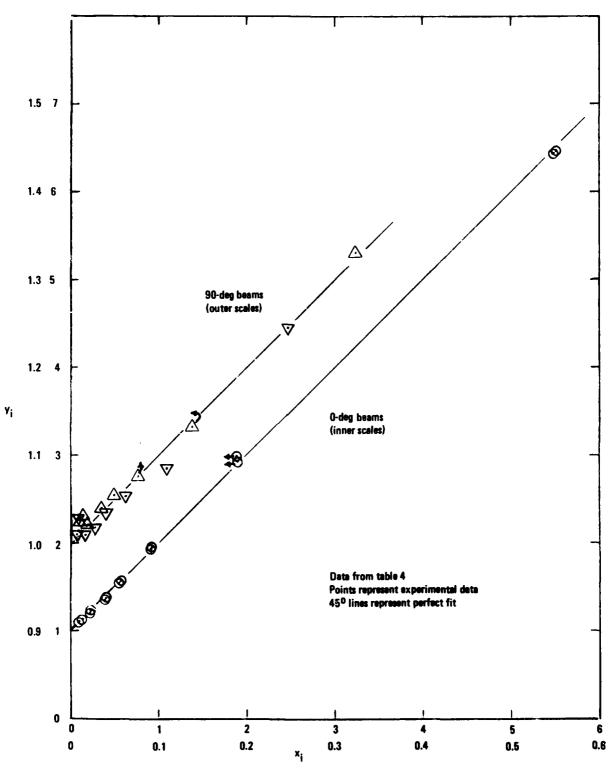


Figure 8. - Deflection Rate Ratios Corresponding to Best Fit Solutions for E  $_{\mbox{\scriptsize XX}}$  and G  $_{\mbox{\scriptsize XZ}}$ 

### Approximate Method for Shear Calculations

Shear deflection occurring in beams is usually estimated by assigning a value to the Timoshenko shear coefficient K in the relation.

$$w_{\text{total}} = \int \int \frac{M}{EI} dx dx + \frac{1}{K} \int \frac{S}{GA} dx$$
 (7)

Here the first term is the Bernoulli-Euler bending deflection found in handbook formulas; it recognizes no restraint of section warping. The second term is a "shear correction" which includes all other effects, presumed to cause a shearing type of deflection. K is generally thought to be a section property that varies only with section shape; various sources cite values ranging from 2/3 to 1.0. Consideration of the shear stress destribution in beams makes it clear that K may vary over the span (Reference 19). Therefore, any value which is to be used outside of the integral as in (7) must be specific not only for load and section but for span.

The results of the current study can be utilized to develop rational values of K. Such data, obtained by introducing the appropriate values from Table 4 into (7) and solving for K, are presented in Figure 9. (For the 90° beams loaded in the 3-direction, the value of E was taken as  $E_{22}/(1-v_{21}v_{12})$  in accordance with representation as plane strain.)

Reissner (Reference 20), and Nair and Reissner (Reference 21), working from fundamental energy considerations, have established upper and lower bounds for the shear deflection that occurs in a beam which does not deform locally at the load and support stations. In the 90-degree beam loaded in the 1-direction, the high fiber stiffness produces boundary conditions which are closely represented by Reissner's model, and the resulting values of K, as plotted in Figure 9, are seen to lie close to the prescribed bounds calculated



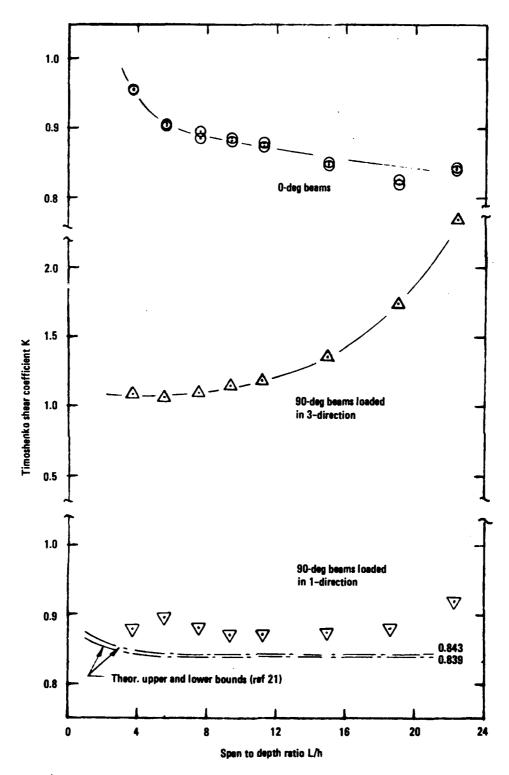


Figure 9. - Variation of the Timoshenko Shear Coefficient with Span and Direction of Load

from Reference 21. When the load is transverse to the fiber, however, local deformation at the loading station is significantly greater, and consequently K for very short spans is increased and may exceed 1.0.

These differences in the value of K are produced by variation in the way shear is introduced into the beam, interacting with section constraint. Shear stress distribution adjacent to the loading station for the 90-degree beam, L = 1.02 inch, loaded in the 3- direction, is contrasted with that for load in the 1- direction in Figure 10 (data from the relaxation solutions). The high peak stress and distorted distribution associated with loading transverse to the fibers tends to produce large section warping. The central section cannot warp, however, and adjustment to this condition reduces overall beam deflection. Thus, short beams with low  $\rm E_{ZZ}$  may appear to be stiffer in "shear" than those for which  $\rm E_{ZZ}$  is large.

Accuracy in the calculation of K is adversely affected by limitations in accuracy of the current set of relaxation solutions. While these data have a precision of better than 0.1 percent, they suffer a small error as a result of the finite dimensions of the grid. Such errors have a greater effect on K than on the shear modulus, because after  $G_{\rm XZ}$  has been evaluated, the relaxation solution values enter once again in the  $w_{\rm total}$  term of Eq. (7), and in this case without the benefit of multiple points. For cases important in design, further investigation is desirable to confirm the results and to establish a broader basis for extrapolation to other materials.



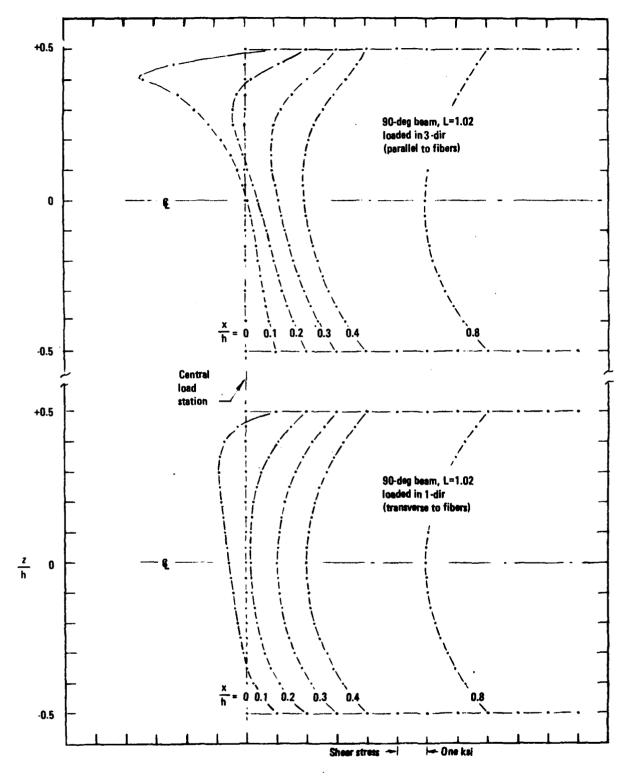


Figure 10. - Calculated Shear Stress Distributions for 90-degreee Beams, L = 1.02-in

## Overall

where data obtained in the axial load tests and the flexure tests overlap, only small differences are seen to exist, and these discrepancies probably represent the accuracy of the test methods. By taking rounded values which are consistent with each other and the indications of transverse isotropy, a complete set of the nine elastic constants is obtained and presented in Table 6.

These data represent the overall results of replicate tests and various test conditions. Scatter of test results with different test procedures is no more than a few percent.

These results were derived by assuming linear, orthotropic elasticity, an assumption justified by the structure of the material and the linearity exhibited in the basic load-deflection data. The fact that the final results provide a complete, self-consistent set of properties confirms this model and appears to refute suggestions of non-linear elasticity relations found in the literature (Reference 16 and 17, for example). However, the scope of the current program with regard to stress range and condition is insufficient to resolve this question. For the type of loading and the material tested, simple orthotropy provides a satisfactory model.



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TABLE 6 OVERALL SUMMARY

Elastic Constants of T300/5208 Laminate As Tested

PROPERTY	VALUE	
	(GPa)	(Msi)
E <sub>11</sub> E <sub>22</sub> , E <sub>33</sub>	145. 10.7	21.0 1.55
G <sub>12</sub> , G <sub>13</sub> G <sub>23</sub>	4.50 3.70	0.65 0.52
ν12 'ν13 ν23 'ν32 ν21 'ν31		0.31 0.49 0.023

## CONCLUSIONS

A complete set of the nine elastic constants experimentally determined for a 64-ply T300/5208 unidirectional laminate is presented in Table 6. These data are based on multiple independent experimental determinations, with the overall means of the test values modified slightly to obtain a consistent set.

The results agree with values cited in the literature, except in the case of the in-plane shear modulus  $G_{12}$ . The value obtained here (0.65Msi) is somewhat lower than that expected from  $\pm$  45° tension tests (0.80 Msi) and the Pagano torsion test (0.87 Msi), and substantially lower than reported for ultrasonic tests (1.03 Msi).

The results obtained confirm, to the limits of the experiment, that the elastic stress-strain relations are linear and that the material is transversely isotropic.

Calculations of the Timoshenko shear coefficient indicate a substantial variation with the orientation of the orthotropic axes of the beam and with the span. In some cases the calculated values exceed by a considerable amount the recognized bounds for a cantilever which has slightly different end conditions. Boundary conditions are thus seen to have an unexpectedly strong influence on the deformation attributable to shear. Such an effect could explain variations in performance associated with different test procedures, and would increase the problems of analysis and design.



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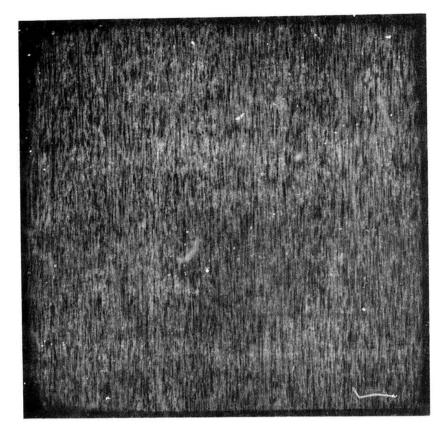
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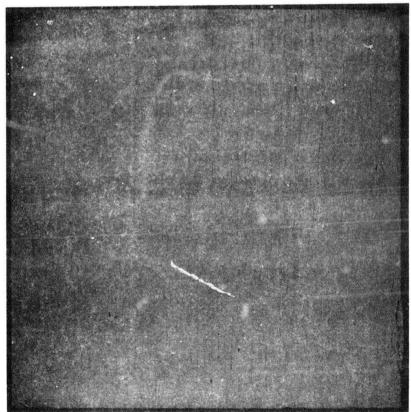
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## APPENDIX

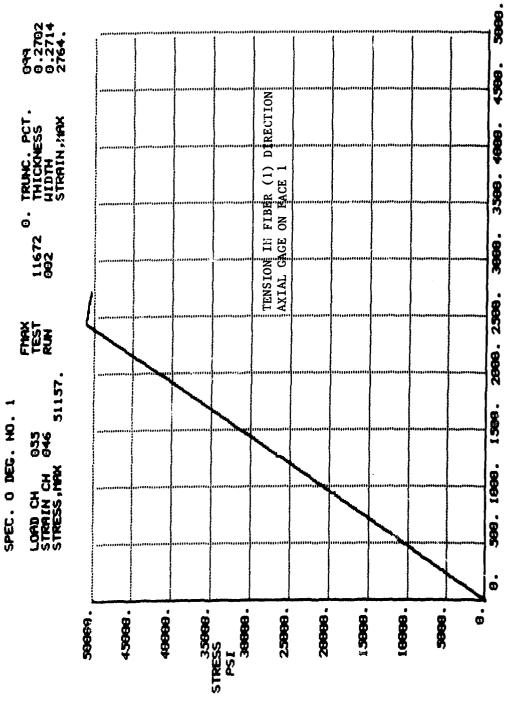
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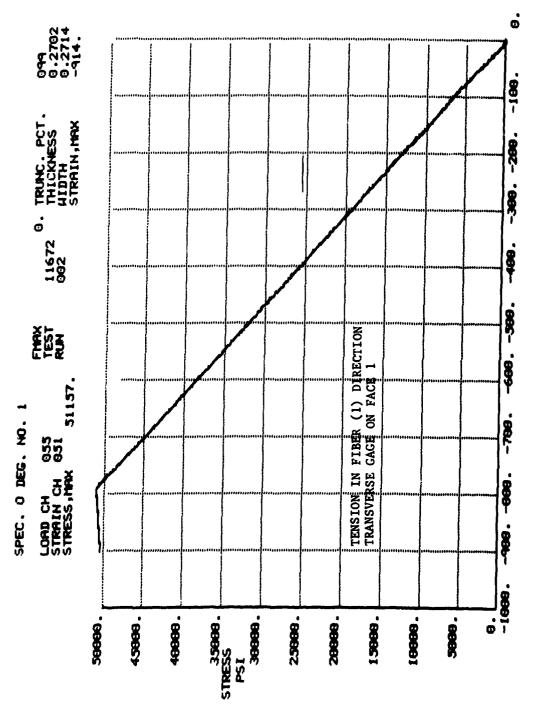


l (left) and  $90^{\rm O}$  No. l (right), at 15x Damage to external surface of 00 specimen was caused by test Figure A-1 - Sections of Flexure Test Specimens 0° No. Magnification. machine grips.

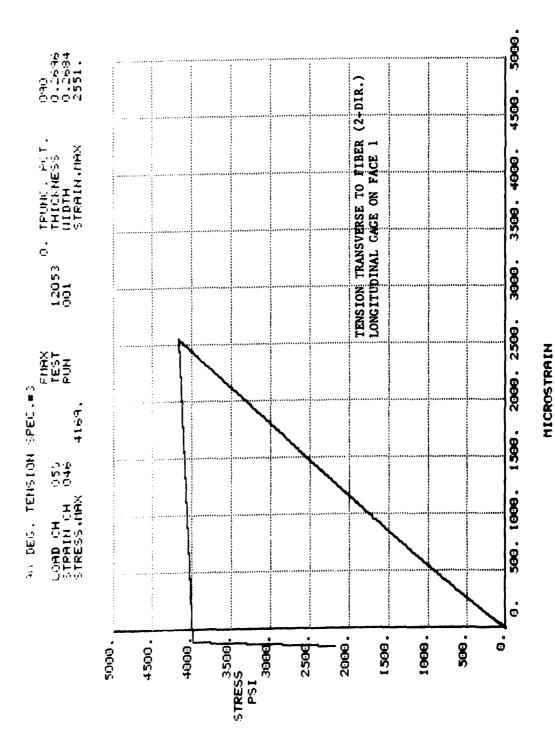
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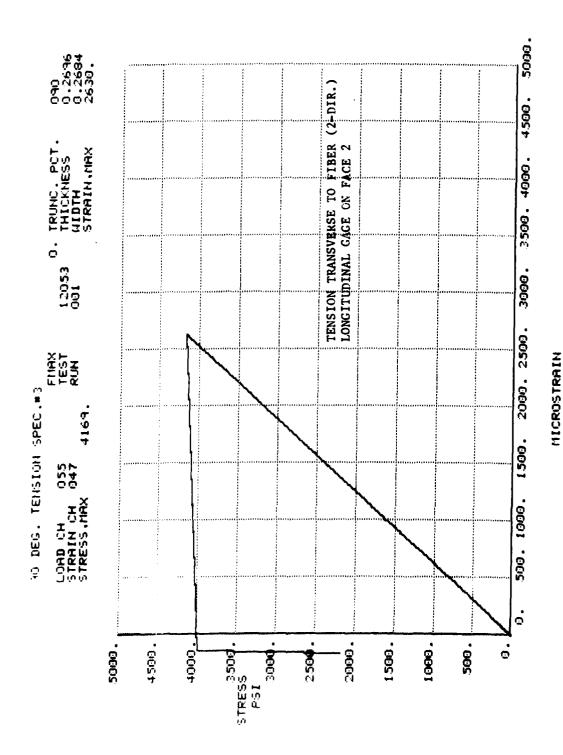


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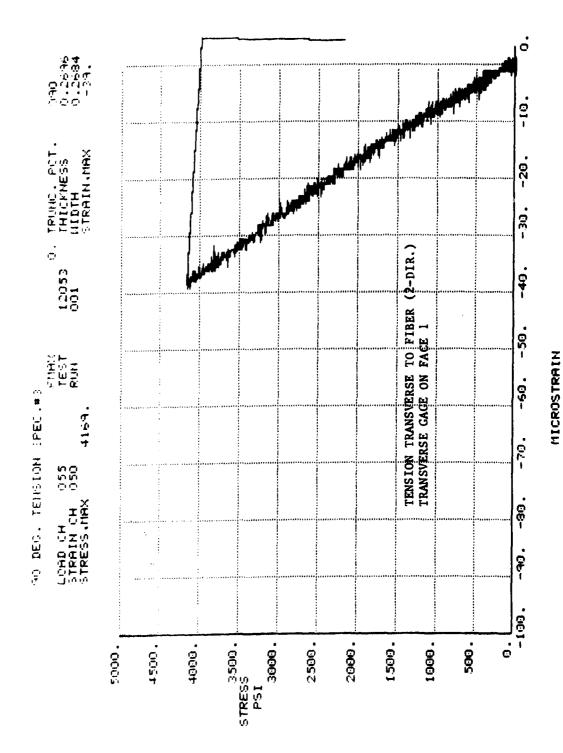


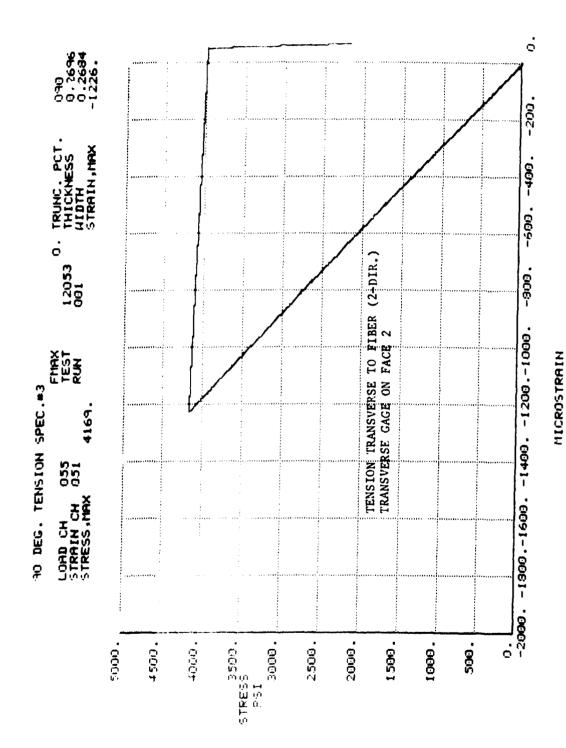
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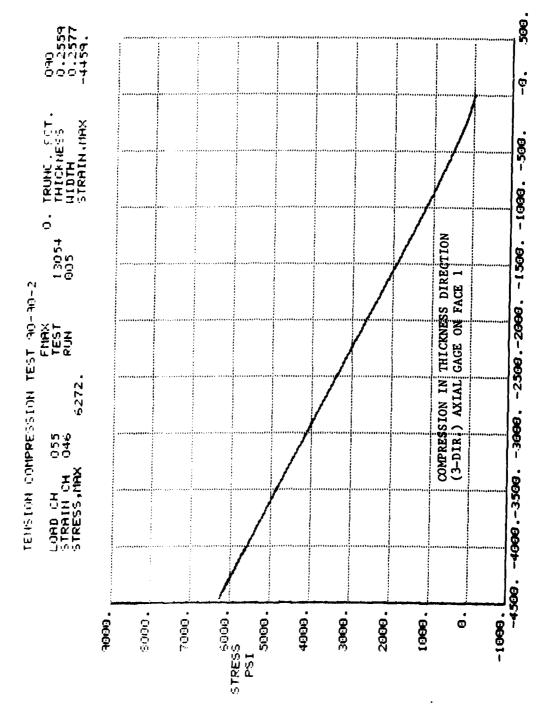




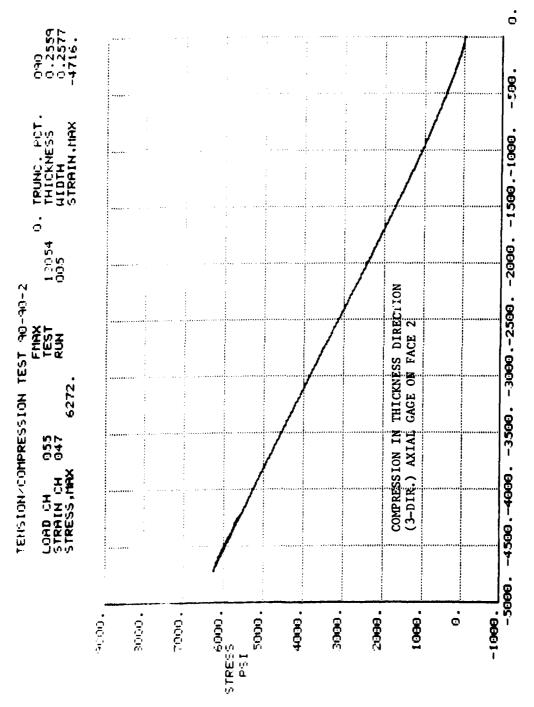
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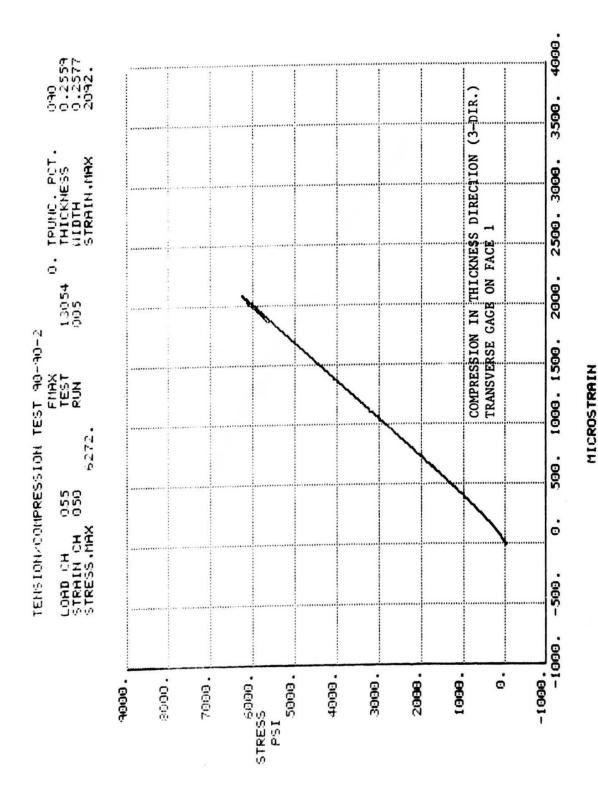




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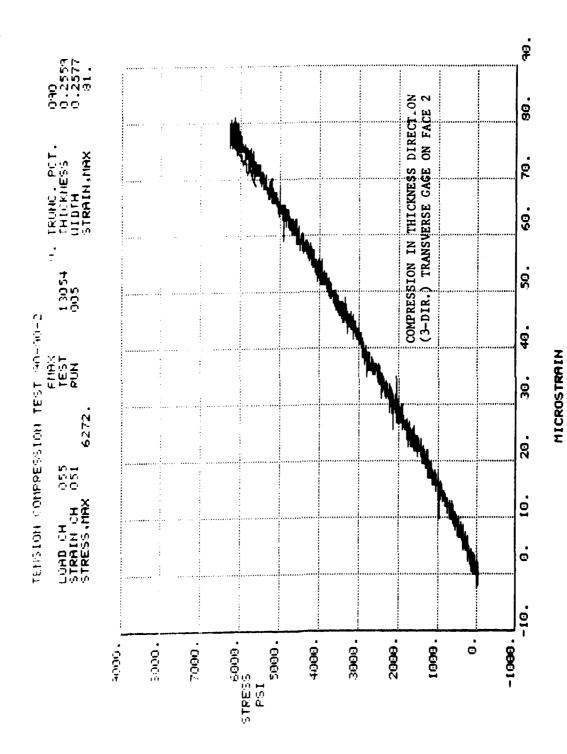


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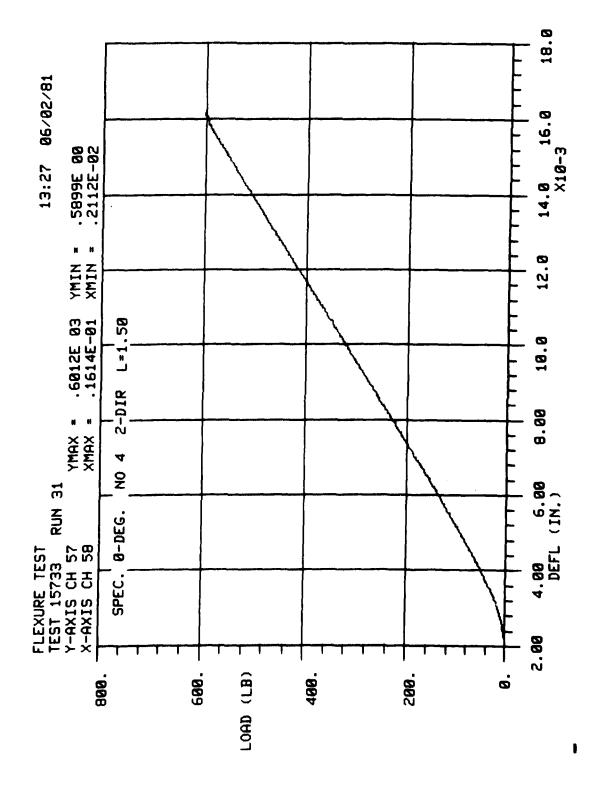


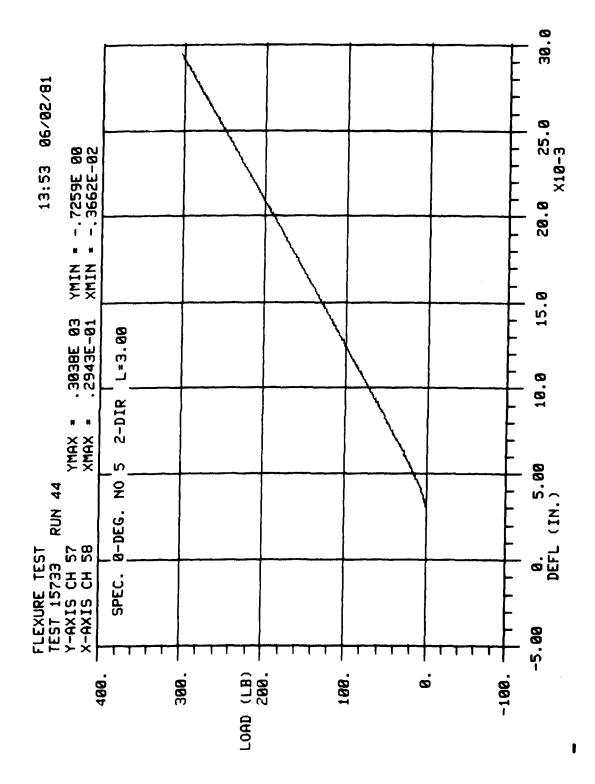
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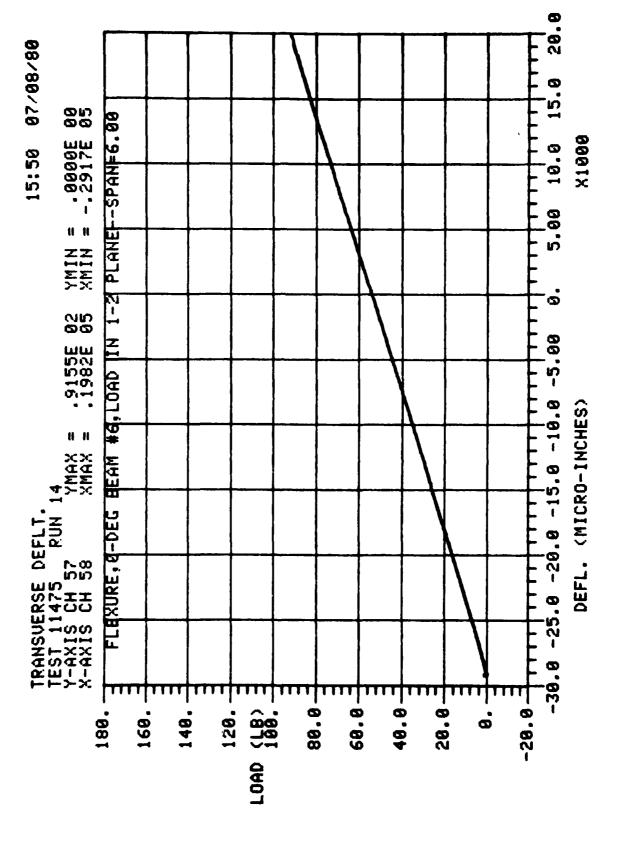
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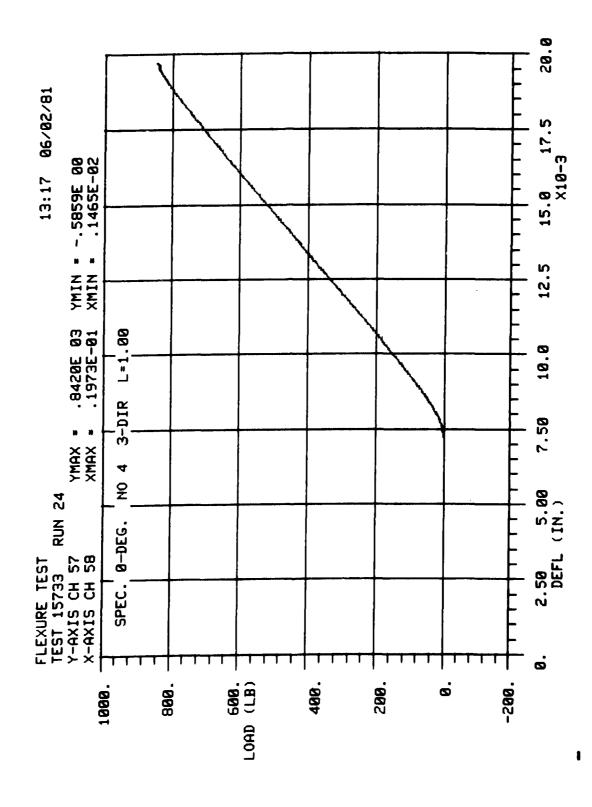


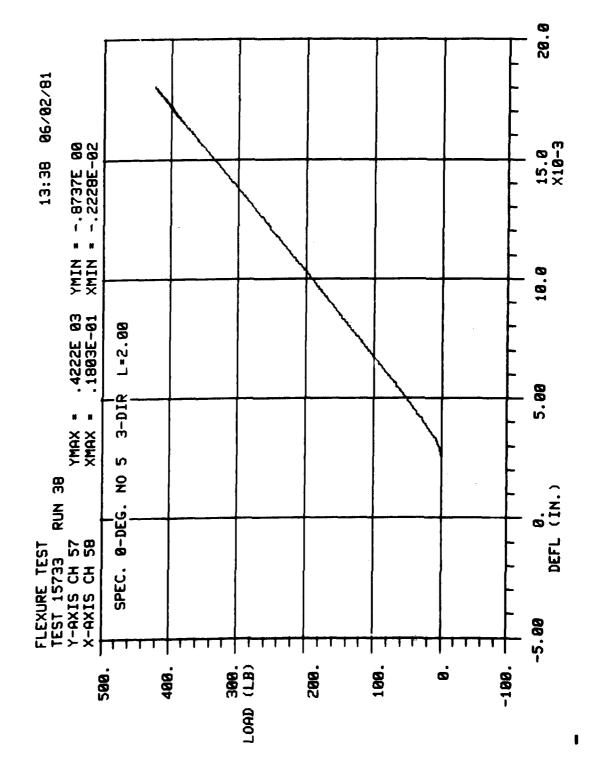
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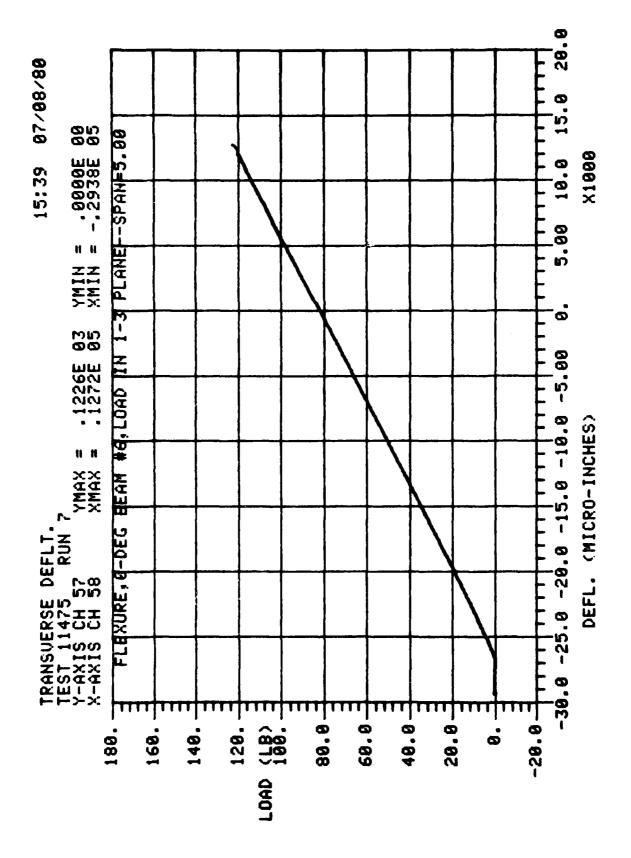


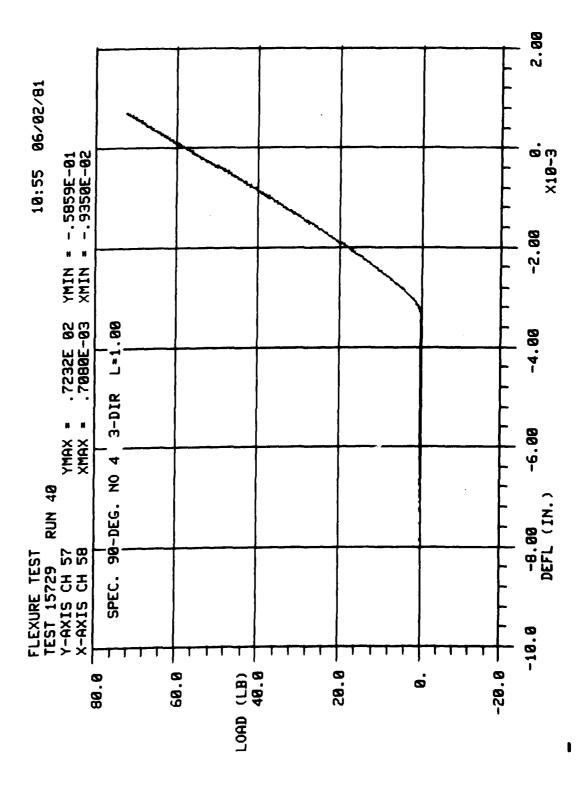


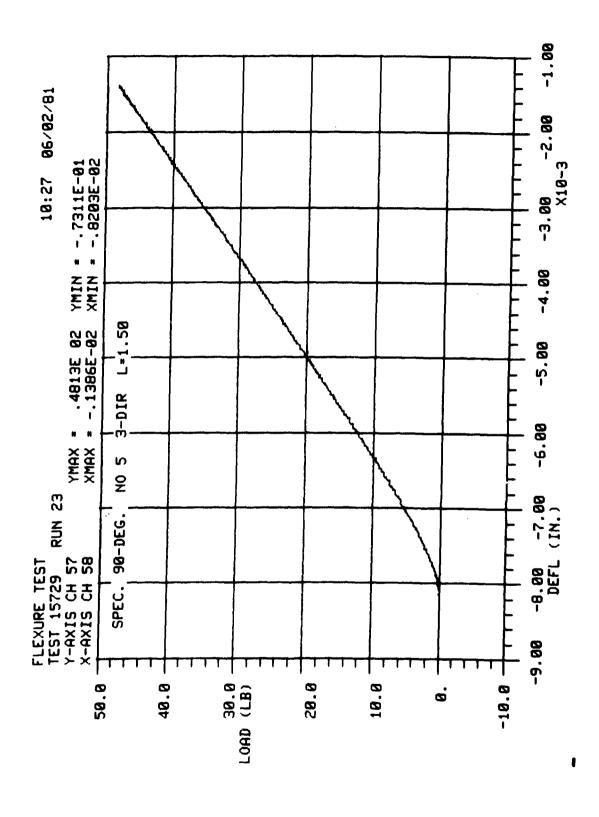


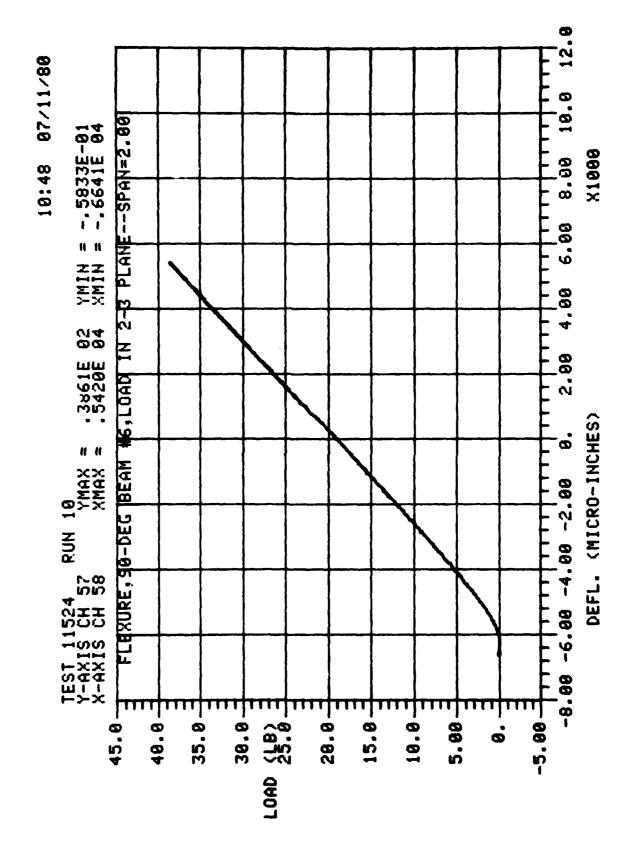


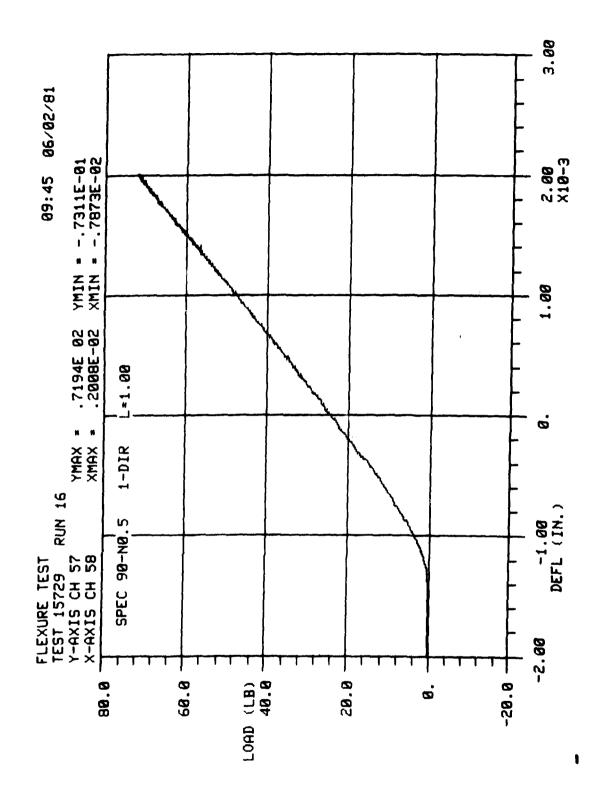


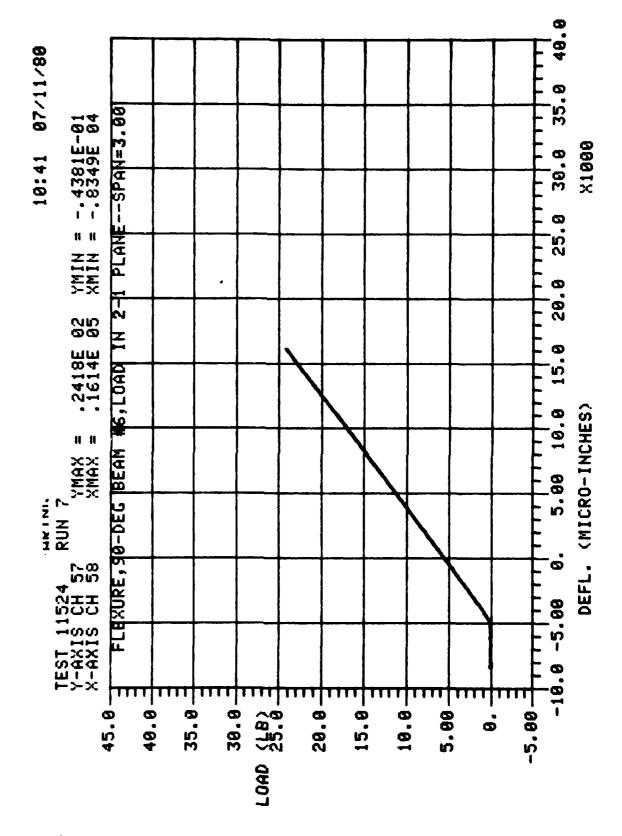


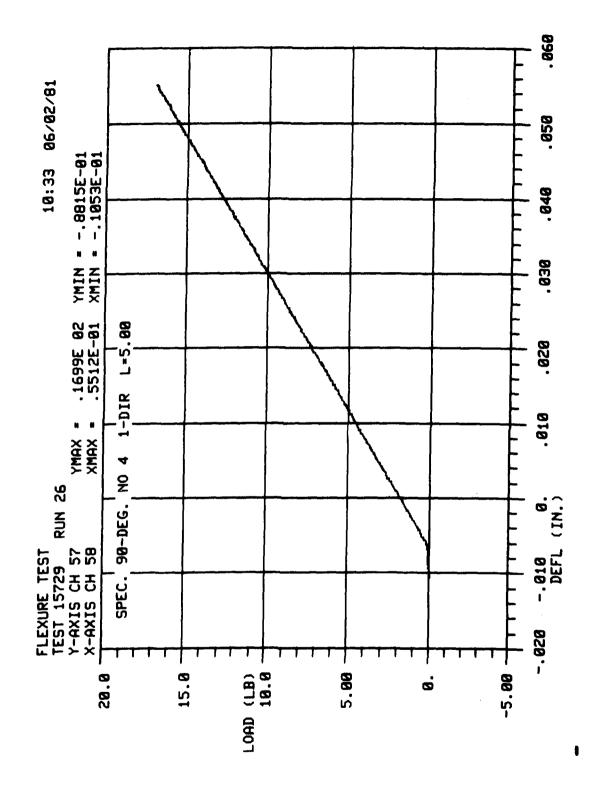




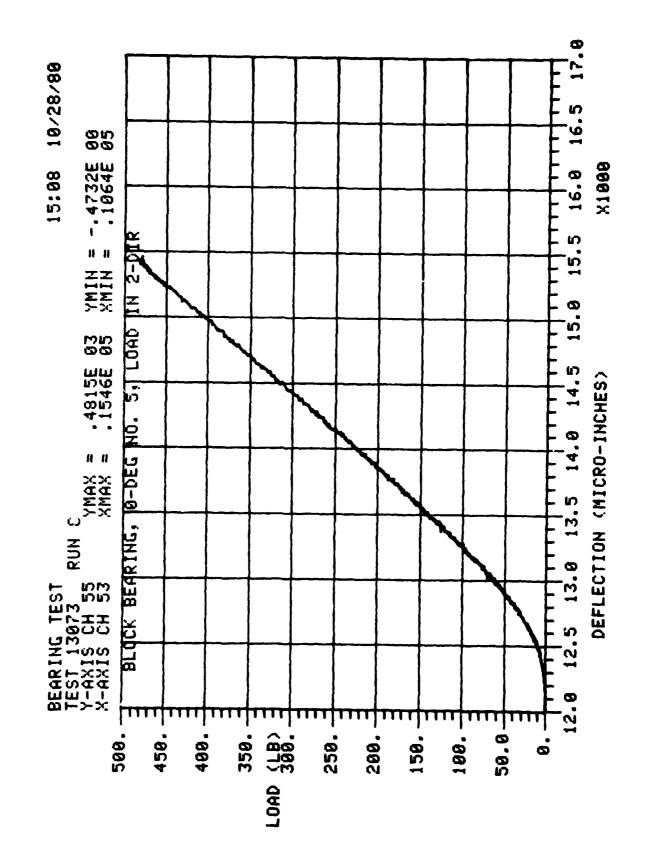


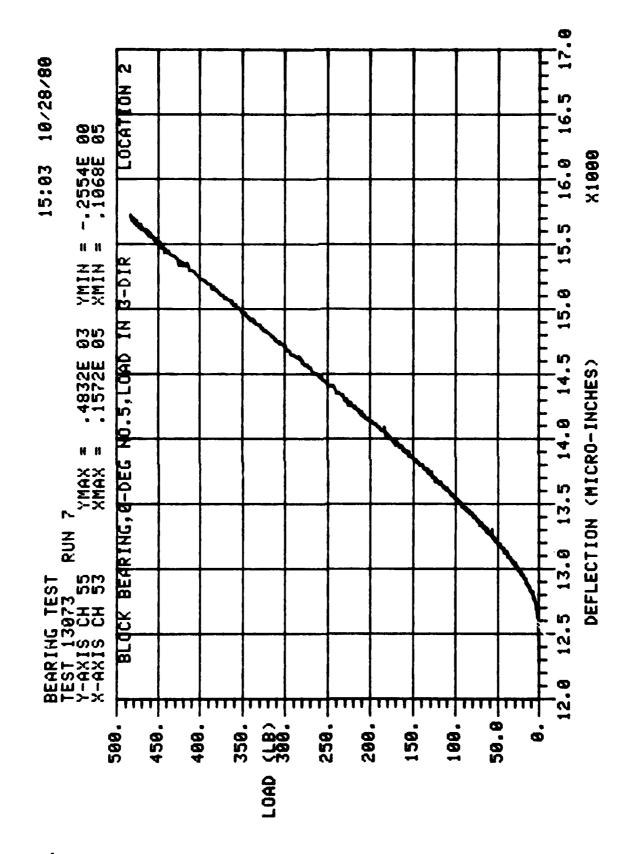


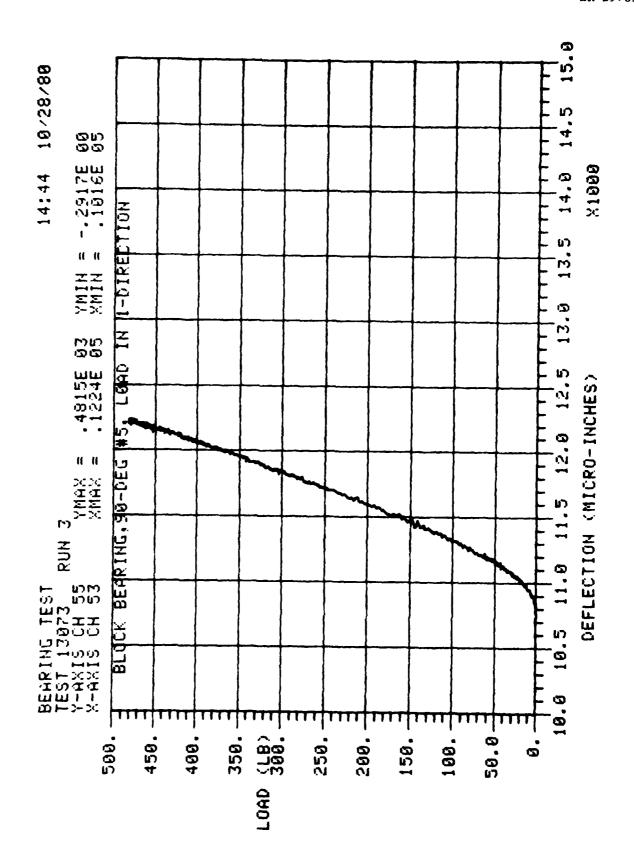


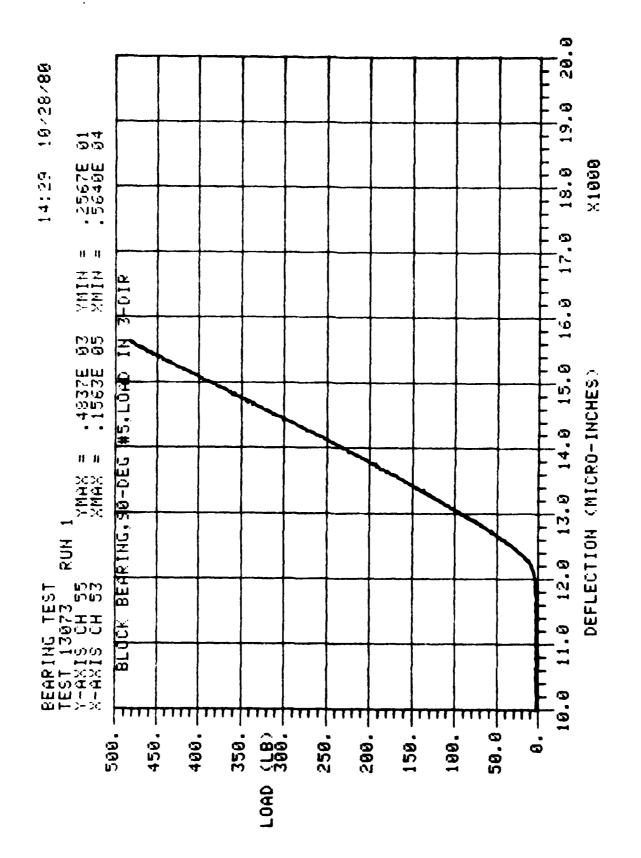


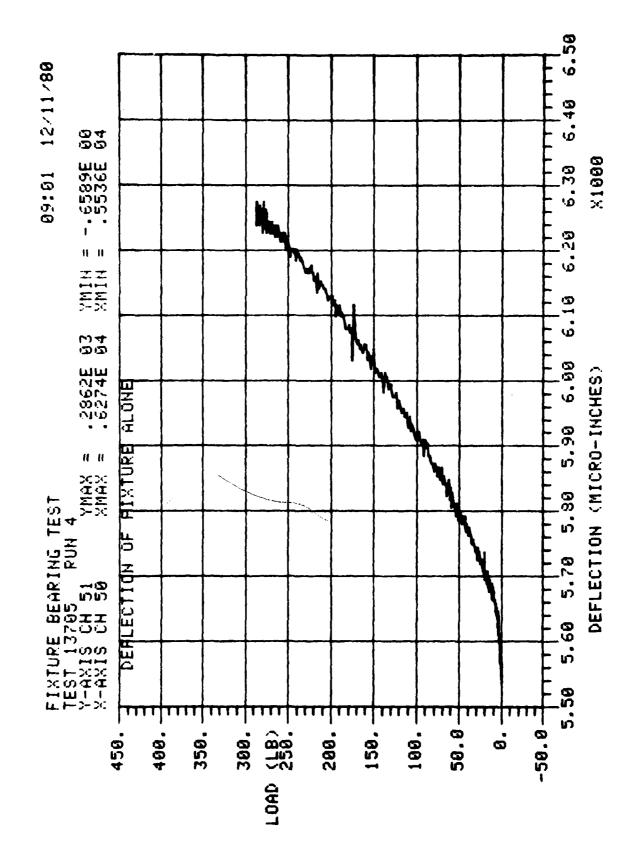




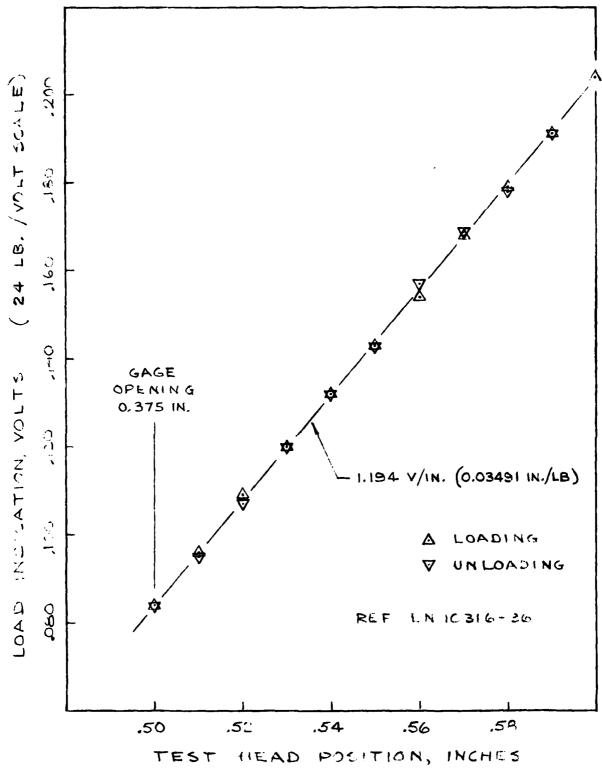








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TABLE A-1. COMPLIANCE COEFFICIENTS

TENSION TESTS IN FIBER DIRECTION (I-DIR.)

STRESS		TEST		COMPLIAN	$CE$ $S_{ij} = \delta$	e; /ðo; (ue/psi)
RANGE	SPECIMEN	NO-	GAGE	3,,	521	S <sub>31</sub>
(ksi)					-21	931
0-10	0° No. 1	11672/2		0.05091	-0.01519	
	٥		2+4	0.05092		-0.01525
	0° No. 2	11767/1	1+3 2+4	0.05/36	-0.01715	-0.01877
	u		İ	0.05022		-0.0/5//
		11767/2	/+3 2+4	0,05192	-0.01699	-0.01801
	0° No. 3	12046/4	ļ	0,04881	-0,01742	
	0 110, 0	720 197	2+4	0.04802	0,0,,42	-0.01566
10-40	0° No. 1	11672/2		0.04834	-0.01488	
			2+4	0.04795		-0.01484
	0° No. 2	11767/1	1+3		-0.01475	8
			2+4	0.04698		-0.01505
	4	11767/2	1+3 2+4	0.04826	-0.01473	-0.01498
	00 4/ 3		İ	l	00.55	0,0,430
	0° No. 3	12046/4	2+4	0.04717	-0.01556	-0.01410
40-50	0° No. 1	11672/2	143	0.04657	-0.01486	1
		1	2+4	0.04553		-0.01481
	0 ° No.2	11767/1	1		-0.01456	
			2+4		_	-0.01504
		1/767/2		0.04710	-0.01445	-0.01497
			2++	0.04621		-0.07437
			L			
10-40	N			8	4	4
	MEAN			0.0461	-0.0150	-0.0147
	COEF. VA	AR'N		0.014	0.027	0.030

TABLE A-2. COMPLIANCE COEFFICIENTS

COMPRESSION TESTS IN FIBER DIRECTION (I-DIR.)

SPECIMEN	TEST	STRESS	GAGE	COMPLIANCE	$= s_{ij} = \partial e_i /$	'δσ; (με/psi
	NO.	RANGE (ksi)		5,,	52,	S <sub>s</sub> ,
0° No.2	11851/4	1-6	/+3 2+4	0.04952 0.04846	-0.01454	-0.01465
0° No. 2	11851/5	1-10	/+3 2+4	0.05018 0.04901	-0.01560	- 0.01583
0° No. 3	12046/2	5-//	/+3 2+4	0.04957 0.04878	- 0.0/530	-0.01390
	N MEAN COEF.	VAR'N		6 0.0492 0.016	3 -0.0151 0.036	3 -0.0148 0.066

TABLE A-3. COMPLIANCE COEFFICIENTS
TENSION TESTS IN-PLANE, TRANSVERSE (2-DIR)

<del>,~~~~~~</del>			·····			
SPECIMEN	TEST	STRESS	GAGE	COMPLIANC	$E s_{ij} = \partial e_i / \partial$	o; (ut/psi)
	NO.	RANGE (ksi)		522	5,2	532
90° No. 1	11676/2	0-3	/+3 2+ <b>4</b>	0. <b>6486</b> 0. <b>6484</b>	-0.01604	-0.3104
90° No. 2	11761/1	0-3	1+3 2+4	0.6397 0.63/2	-0.01563	-0.3056
	11761/2	0-6	1+3 2+4	0.6537 0.6575	-0.01559	-0.3078
	11763/1	0-6	/+3 2+4	0.6386 0.6420	-0.01541	-0.3049
	11763/2	0-5	/+3 2+4	0. <b>635</b> 1 0.6360	-0.01539	-0.3067
90° No. 3	12053/1	0-4	/+3 2+4	0.6385 0.6417	-0.01527	-0.3050
<i>N</i>				12	6	6
					]	
	MEAN	,		0.643	- 0.0156	-0.307
	COEF	. VARN		0.012	0.036	0.007



TABLE A-4. COMPLIANCE COEFFICIENTS

COMPRESSION TESTS IN-PLANE, TRANSVERSE (2-DIR)

SPECIMEN	TEST	STRESS	GAGE	COMPLIANCE	ε s <sub>ij</sub> = δe <sub>i</sub> /δ	g (ue/psi)	
	NO.	RANGE (ksi)		522	S, 2	S <sub>32</sub>	
90° No. 1	11676/5	1-2	1+3 2+4	0,6276 6.6341	-0.01473	-0.3022	
	11676/6	1-3	1+3 2+4	0.6413 0.6369	-0.01373	-0.29/8	
50° No. 2	12037/2	1-3	1+3 2+4	0.6159 0.6124	-0.01544	-0.2967	
				_	_		
	N			6	3	3	
	MEAN COEF.	VAR'N		0.628	0.059	-0.297 0.018	

TABLE A-5. COMPLIANCE COEFFICIENTS

COMPRESSION TESTS IN THICKNESS DIRECTION (3-DIR)

SPECIMEN	TEST NO.	STRESS RANGE	GAGE	COMPLIA	i/do; (ue/psi)		
	NO.	(ksi)		5 <sub>33</sub>	5,3	S <sub>23</sub>	
0° No. 2	12032/5	5 - 10	2+4	0.6608	-0.0175		
(5-TIER)	12032/6	6-19	2+4	0.6722	-0.0178		
90° No. 2	12055/2	<i>5-</i> 9	2+4	0.6633		-0.3522	
(3-TIER)	12055/3	2-12	2+4	0.6658		-0.3591	
	12058/5	8-18	2+4	0.6602		-0.3589	
90-90-2 (17-TIER)	/3054/4	/-3	/+3 2+4	0.6705 0.6878	-0,0161	-0.3330	
	13054/5	2-6	/+3 2+4	0.6752 0.679 <b>4</b>	-0.0153	-0.3274	
	<i>N</i>		<u> </u>	9	4	5	
	MEA	AN.		0.671	-0,0167	-0.346	
	COE	F. VARN	•	0.014	0.072	0.043	

546204 543503 5536270 5525862 5525862 56264 4955044 455044 455044 455044 455044 455044 455044 436133 3377236 3377236 3377236 3377236 3377236	0.268079 0.252731 0.981735 0.962789 0.962789 0.912674 0.8869666 0.860066 0.887121 0.727646 0.727646 0.548189 0.548189 0.548189 0.548189 0.548189
ဝင်ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ <b>ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခဲ့ခ</b>	
0.474076 0.474820 0.458655 0.458665 0.4367826 0.410886 0.3767886 0.376715 0.376488 0.376488 0.376488 0.376488 0.376488	0.2132228 0.21498228 0.9473228 0.9473228 0.947328 0.856604 0.873284 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228 0.765228
0.40824 0.406824 0.3943549 0.36738257 0.36738257 0.36738257 0.36738257 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582 0.280582	0.4849420 0.594443420 0.694434344443434444343444343444343444343444343
0.340907 0.339358 0.335037 0.328604 0.316922 0.392888 0.292888 0.293888 0.2733028 0.2733079 0.253054 0.253054 0.253054 0.253054 0.253054 0.253054 0.253054 0.253054	
0.275964 0.274707 0.255889 0.255889 0.255406 0.256373 0.226373 0.226373 0.226373 0.178408 0.187200 0.187200 0.187200 0.187200 0.187200 0.187200 0.187200	paaaaacrrraaaannnutaaa
0.214591 0.213595 0.216787 0.195536 0.182977 0.182977 0.176517 0.176517 0.176517 0.176517 0.176517 0.177715 0.137715	0.094980 0.094980 0.094980 0.853039 0.853039 0.774258 0.774258 0.774259 0.754259 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.656943 0.6693 0.6993 0.69
0.157991 0.157231 0.157231 0.157884 0.1588493 0.1788693 0.1788693 0.1788693 0.1788693 0.1788693 0.1788693 0.1788693 0.1788693 0.0955604 0.0996604 0.0996604	0.068700 0.068707 0.068707 0.786870 0.786870 0.7886370 0.681725 0.
0.106937 0.106937 0.106937 0.103061 0.0900198 0.093664 0.093664 0.093664 0.093664 0.0936693 0.06093595 0.060917	0.048516 0.045731 0.745385 0.740619 0.743750 0.6574134 0.6574134 0.6574134 0.5574738 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288 0.5751288
0.064549 0.064172 0.064172 0.0641688 0.0674748 0.0574748 0.0574748 0.0574774 0.0574777 0.0574777 0.047178 0.047178 0.037491	0.028327 0.026713 0.026713 0.026713 0.0677590 0.0570944 0.0570993 0.058714
0.030890 0.030695 0.030695 0.020335 0.0272874 0.0272874 0.021007 0.021007 0.021007 0.01007 0.011007 0.011007	
0.008529 0.0088295 0.008462 0.008275 0.0087343 0.006678 0.006678 0.006515 0.008515 0.0084985 0.008489	0.003358 0.003188 0.003188 0.545204 0.5452503 0.513411 0.4495646 0.44552446 0.4550444 0.4550444 0.377234 0.377234 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853 0.345853
00000000000000000000000000000000000000	0.000000 0.000000 0.000000 0.474076 0.47826 0.478826 0.478826 0.4788826 0.4788826 0.478888 0.374638 0.374638 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749 0.376749
0.004472 0.004873 0.008682 0.008822 0.007250 0.007250 0.006898 0.006694 0.006694 0.006694 0.006992 0.006993 0.006993 0.006993 0.006993	0.003415 0.003408 0.003408 0.003408 0.303430 0.3034337 0.33434 0.33629 0.33629 0.33629 0.3268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607 0.268607
T	. UU . O - U M 4 N 4 L M 4 O + C M 4 N 4 L M 4 D 4 C M 6 C M 6 C M 7 M 7 M 7 M 7 M 7 M 7 M 7 M 7 M 7 M

820106

Uzy= .49

ee= .027 bb= .013455 VxY= .31

Vxz = .31 P = 100 10 = 0

Grz= 650000. Sh= 185,529 Lp= .0135

Exx= 21100000. Eyy= 0 Ezz= 1550000. PLANE STRESS: Xx= .062811 , Xz= .85504 L= 2.024 h= .2691 W= .2695 L1= .54

0-DEGREE BEAM LUADED IN 2 DIR

FN IN FEBRIN 20

		-1.7704 -2.0891 -2.2191 -2.2907 -2.2907 -2.0463 -1.6858 -1.52889 -1.52889		16669 16669 16669 16600 1.	
		0.6990 0.7941 1.0581 1.0581 1.0581 1.0581 1.0595 0.0850 0.08527 0.08927		1.3909 1.16099 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999 1.00999	
	820106	0.3634 0.2951 0.1765 0.07051 0.0051 0.0836 0.0337 0.0223 0.0327 0.0644		-1.1622 -0.7287 -0.3023 -0.0391 -0.0159 -0.0227 0.0227 0.0239	
	4	1.4361 1.2878 1.1921 1.160 0.8917 0.8290 0.8290 0.8285 0.7548		0.0253	
	Uzy=	2.5405 2.4195 2.4196 2.3348 2.0454 1.8784 1.6964 1.6964 1.5425 1.5168		0.0295	
	Uxy = .31	3.7023 3.6710 3.5710 3.5710 3.4376 3.4446 2.8888 2.5735 2.38834 2.38834 2.38834 2.38834	-37.6532 -21.6170 -16.8270 -13.9294 -12.0789 -10.4926 -10.4926 -10.4926 -10.4926 -10.4926 -10.4936 -10.4936	0.03303 0.0333 0.0333 0.0324 0.0330 0.0330 0.0330 0.0330	43.7429
ANN 20	Uxz= .31 = 100 = 0 = a==	4 9500 4 9 9500 4 9 9505 6 9505 6 9505 6 9505 6 9505 7 9605 7 9605 7 9605 7 9605 8	13.9505 16.2824 13.42824 13.42887 11.1113 19.4924 17.2693 17.2693 16.8993	0.05127 0.05428 0.0529 0.0049 0.0026 0.0206 0.0206 0.0206	1.3747 1.3747 1.0.3747 1.0.0001 1.0.0001 0.00001 0.00001
FN IN FFRANK	= 650000. = 185.529 F: = .0135 ip:	6.3190 6.2949 6.2327 6.1485 6.9524 6.9524 7.9524 7.9524 7.9524 7.1989 7.9523 7.9523 7.9523 7.9523 7.9523	-9.9452 -11.9344 -11.9344 -11.8892 -11.48892 -9.8684 -9.4443 -6.7881 -6.4601 -6.1318	0.0267	-6.9393 -1.9811 -0.6797 -0.61312 -0.0089 -0.0019 0.0018
•	9, 2, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4, 4,	7.8540 7.76298 7.6698 7.5544 7.5544 6.1755 5.3070 7.8695 8.4695 4.6005	-8.0039 -9.2477 -9.2477 -9.5555 -9.5238 -9.5238 -7.3518 -5.9349 -5.9344	-0.2816 -0.2577 -0.2013 -0.1375 -0.0827 0.0070 0.0174 0.0224 0.0223	-5.1046 -2.1268 -0.18857 -0.2092 -0.0050 0.0023 0.0030 0.0030
L= 2,02	1550000. 2= .85504 L1= .54	9.6190 9.5916 9.51916 9.3963 9.2460 8.4722 7.3795 5.655 5.6873 5.0860	-6.5208 -7.2882 -7.6873 -7.7624 -7.7624 -7.1219 -5.2454 -5.2454 -5.0773 -4.8375 -4.5554	-0.1840 -0.1681 -0.1239 -0.0845 -0.0040 0.0128 0.0145 0.0145	.3.8876 .2.0652 .2.0652 .0.5920 .0.0329 .0.0012 0.0046 0.0046
IN 2-DIR	0 Ezz= 62811 , N W= .2695	11.7181 11.54810 11.5781 11.4076 11.1917 11.1917 11.1917 12.130 6.8623 6.8623 6.8623 6.1651	-5.2156 -5.6963 -6.0395 -6.2017 -5.8080 -5.1523 -4.2203 -4.0245 -3.6244	0.0037 0.0037 0.0037 0.0037 0.0037 0.0037 0.0037 0.0037 0.0037 0.0037 0.0037	-3.0518 -1.9121. -1.0297. -0.3105 -0.0111 -0.0111 -0.0057 0.0057
M LOADED	9. Eyy:   Kx= .(   .2691	14,3451 14,13823 14,1036 13,8257 13,4748 13,4748 13,4748 1,3573 1,7311 7,7311 7,7313 6,9282	-4,0002 -4,3170 -4,7459 -4,7459 -4,7955 -4,6869 -3,3683 -3,3683 -3,2145 -3,0577	1 tresses a -0 0.314 -0 0.334 -0 0.334 -0 0.334 -0 0.034 -0 0.004 -0 0.004 -0 0.0034 -	-2.4548 -1.7249 -1.0157 -0.5749 -0.3347 -0.0126 0.0035 0.0075 0.0095
DEGREE BEAI	1110000 STRESS 24 h	stress: 17,9462 17,9462 17,7803 17,4189 16,1865 16,1865 11,1387 9,0203 8,5008 8,5008 8,5004 7,6094	-2.8679 -3.0701 -3.0701 -3.5024 -3.5024 -3.3885 -2.3525 -2.5035 -2.5035 -2.5035 -2.5035 -2.5035 -2.5035	15.Ver 5e 5 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000 0.0000	-2.0106 -1.5312 -0.5683 -0.5788 -0.0489 -0.0128 -0.0107 0.0131
0-D	Exx# 2	A - 4 M 4 B 4 4 7 5 8 9 9	0-0W48547E	7 0 - 2 2 4 8 4 3 7 8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9	0-0m4m04cm60

0.00160280

Rend defl wb

0,0000 0,0000 0,1748 0,1748 0,1748 0,1748 0,4757 0,4289 0,6412 0,473 0,5821 0,5821 0,5821 0,5828 0,3846 0,5578 0,3744 0,5348 0,53744 0,5375 0,0000 0,0000 0,1375
1.1730 1.0286 0.9057 0.6568 0.6486 0.6485 0.6517 0.6553
943mma 0.00000000 0.000095181 0.0001518169 0.0013878 0.0013878 0.00145738 0.00145738 0.0014567 0.0014567 0.00150804 0.001558084 0.00155808 0.00155808 0.00155808 0.00155808 0.00155808

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16	J. L. Har M.I.T. "	х		х			
17	E. Reissner U.C. San Diego "	x		x			
18	A. M. James 76-23 63 A-1		x	x			

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